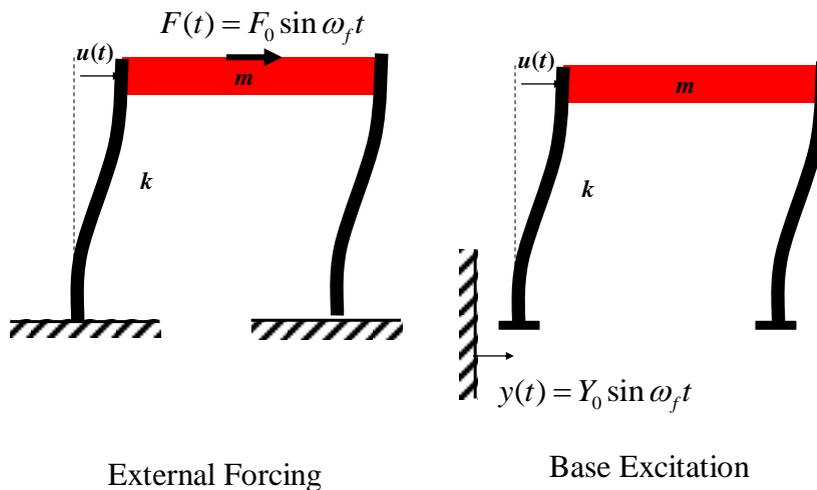
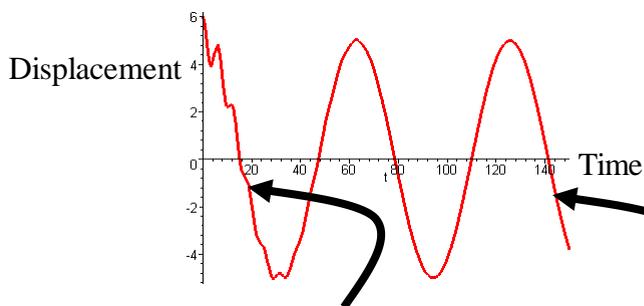


Forced Vibration (Periodic)



Typical Response



Transient vibrations $u_h(t)$ at the natural frequency. Depend on I.C. These decay exponentially with time.

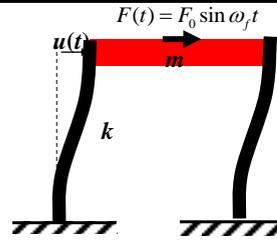
Steady-state Vibrations $u_p(t)$ at the forcing frequency Independent of I.C. These do not decay.

$$\text{Total response: } u(t) = u_h(t) + u_p(t)$$

Amplitude of the steady-state vibrations is (very) large if the forcing frequency is at or near the system natural frequency.

The Steady State
Response:
Undamped

$$\frac{d^2u}{dt^2} + \omega_n^2 u = \frac{F_0}{k} \sin(\omega t)$$

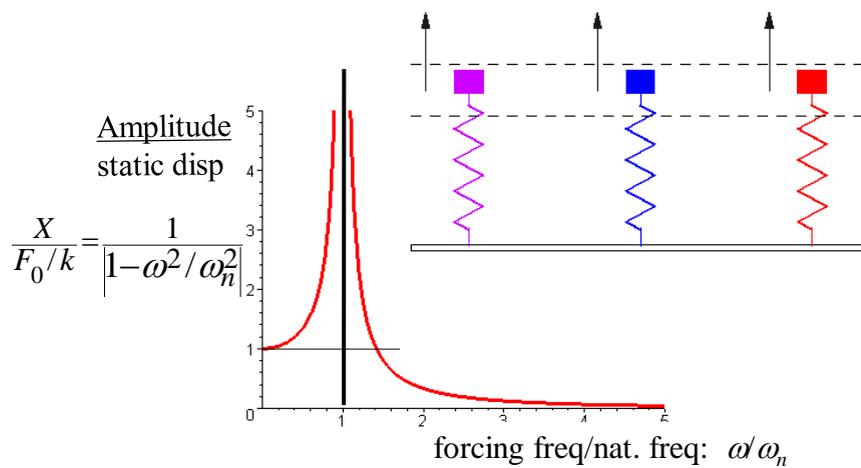


$$u(t) = X \sin(\omega t) \quad X = \frac{F_0/k}{|1 - \omega^2/\omega_n^2|}$$

Amplitude $X \rightarrow \infty$ as $\omega \rightarrow \omega_n$



The Steady State Response



The Steady State Response: Damped

$$\frac{d^2u}{dt^2} + 2\zeta\omega_n \frac{du}{dt} + \omega_n^2 u = \omega_n^2 \frac{F_o}{k} \sin(\omega t)$$

$$u_p(t) = X \sin(\omega t + \phi)$$

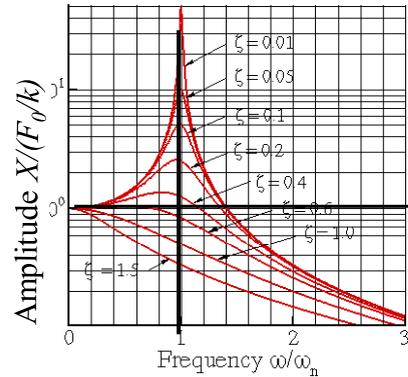
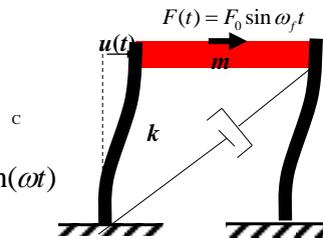
$$X = \frac{F_o/k}{\left\{ \left(1 - \omega^2/\omega_n^2\right)^2 + \left(2\zeta\omega/\omega_n\right)^2 \right\}^{1/2}}$$

$$\phi = + \tan^{-1} \frac{-2\zeta\omega/\omega_n}{1 - \omega^2/\omega_n^2}$$

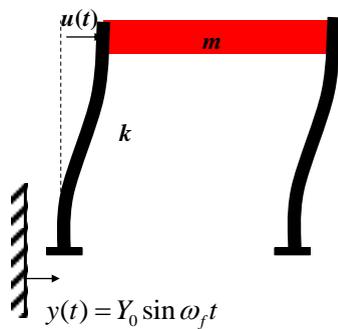
Slow forcing: $X \rightarrow F_o/k$ for $\omega \ll \omega_n$

Resonance: X gets large as $\omega/\omega_n \rightarrow 1$

Fast forcing: $X \rightarrow 0$ for $\omega \gg \omega_n$



Base Excitation: Undamped



$$m(\ddot{u} - \ddot{y}) + ku = 0$$

$$\frac{d^2u}{dt^2} + \omega_n^2 u = \frac{ky}{m} = Y_o \omega^2 \sin(\omega t) = \frac{F_o}{m} \sin(\omega t)$$

$$F_o = mY_o \omega^2 = kY_o \omega^2 / \omega_n^2$$

$$X = \frac{F_o/k}{\left|1 - \omega^2/\omega_n^2\right|} = \frac{Y_o \omega^2 / \omega_n^2}{\left|1 - \omega^2/\omega_n^2\right|}$$

Amplitude of the steady state response

$$X = \frac{F_0/k}{|1 - \omega^2/\omega_n^2|} = \frac{Y_0 \omega^2/\omega_n^2}{|1 - \omega^2/\omega_n^2|}$$

