

2) Analyzing deformation of solids with FEA

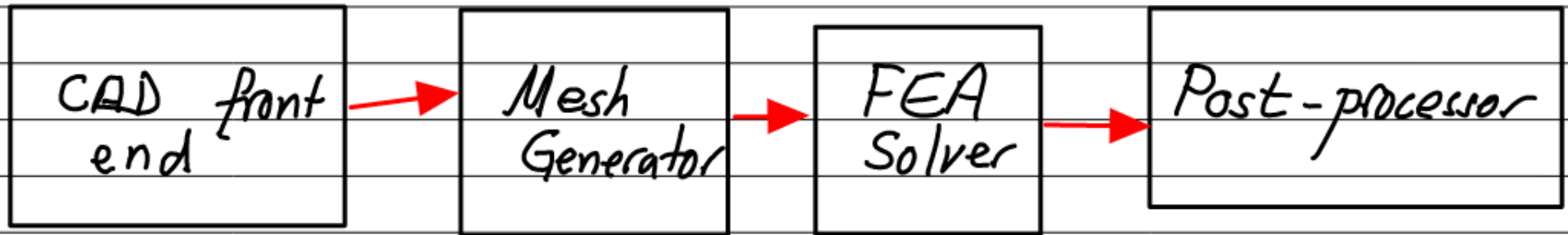
Focus: High level Overview: Derive eqs later

FEA is a general procedure for solving PDEs

In Solids, we solve :

- (1) Strain - Displacement eq
- (2) stress - strain law
- (3) Equilibrium or $\underline{F} = m\underline{a}$

General structure of FEA



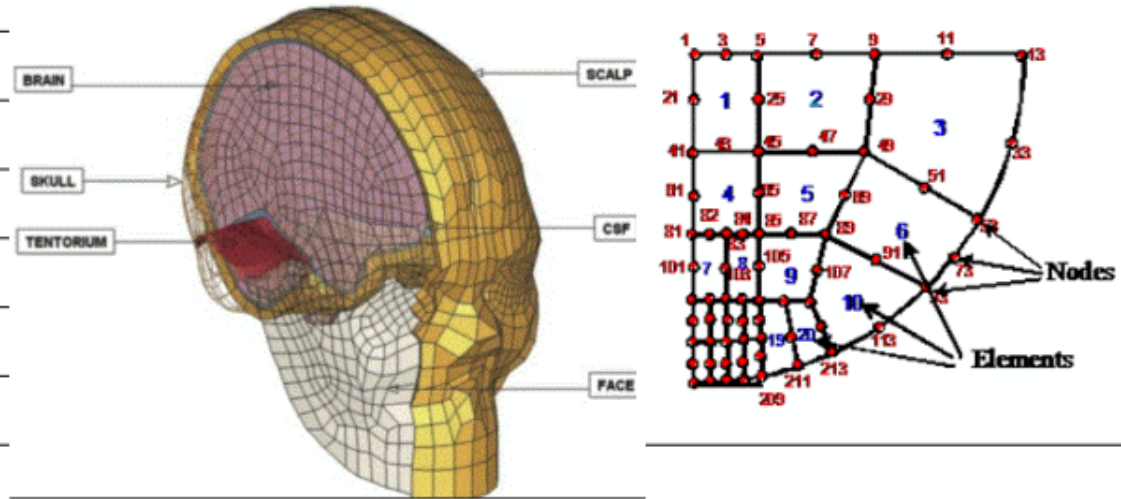
Specifying an FEA problem

- (1) Geometry - Mesh
- (2) Material Model
- (3) Specify Loading : (Contacts)
- (4) Analysis technique

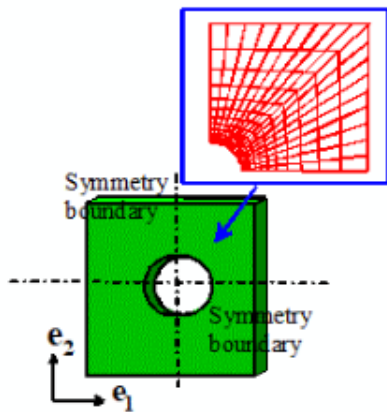
2.1 FEA mesh

3D: Sub-divide solid into cells

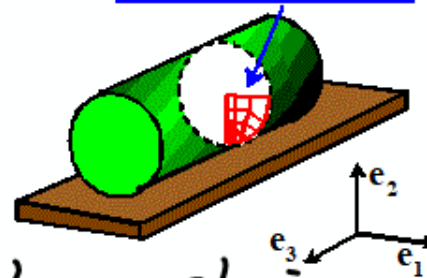
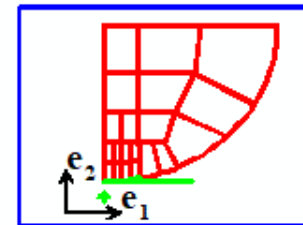
2D: Sub-divide area



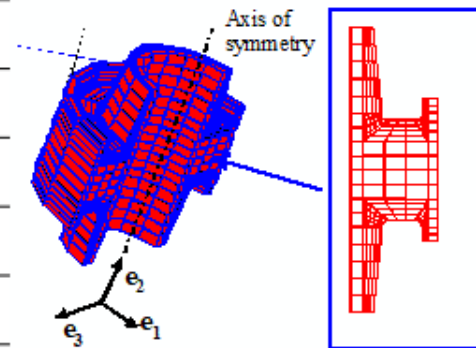
Types of 2D mesh



Plane Stress: Solid is thin sheet loaded in-plane



Plane strain
Solid is long cylinder
No axial stretch



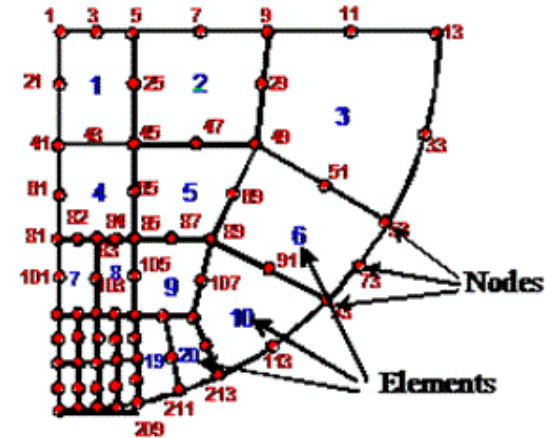
Axisymmetry

Features of a mesh

Nodes : Discrete points in solid

Displacement vector will be calculated at each node

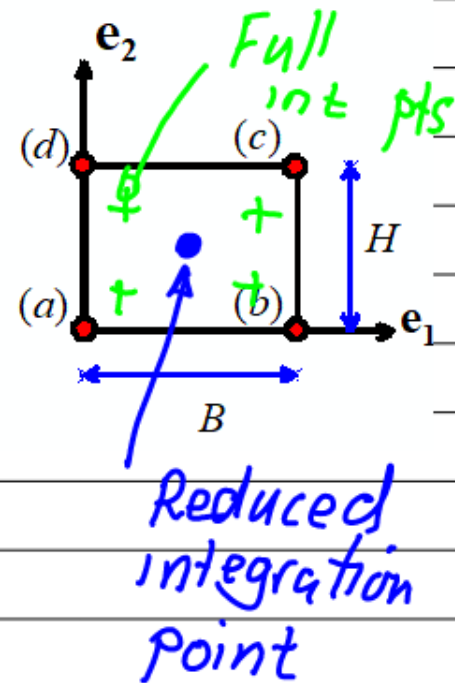
(sometimes also rotations)



Elements : Sub divide solid

Properties: (1) Connectivity (list of nodes on element)

(2) Interpolation scheme : specifies displacement inside element



example: for 2D rectangle : $\xi = X_1/B$ $\eta = X_2/H$

$$\underline{u} = \underline{u}^a (1-\eta)(1-\xi) + \underline{u}^b \xi(1-\eta) + \underline{u}^c \xi\eta + \underline{u}^d \eta(1-\xi)$$

Linear : ABAQUS also has quadratic elements

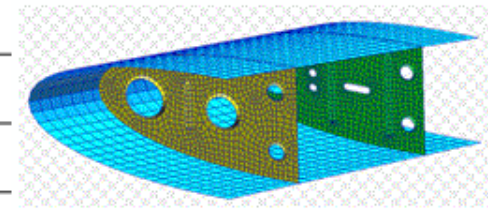
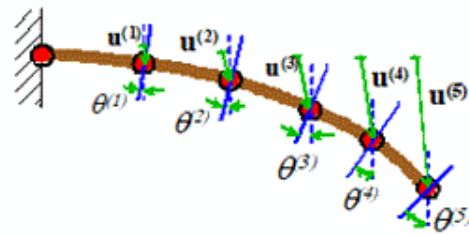
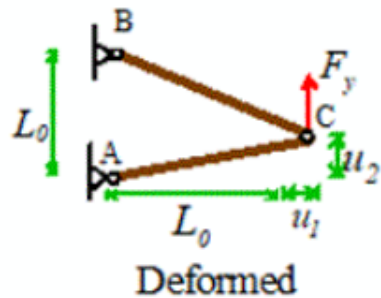
(3) Integration Points : special points inside element

Stresses & strains calculated @ int pts

"Reduced" integration - uses fewer points
- needed for near incompressible materials

Special Element Types

For some solids with special shapes
we can simplify stress / strain fields



Truss element :
 can stretch or rotate
 Specify x-sect area

Beam element
 Elements can bend stretch and twist
 Nodes have rotations and displacements
 Specify x-sect geometry

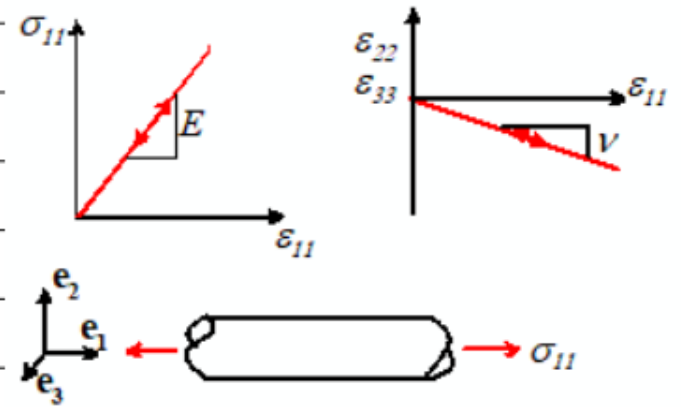
Shell / Plate elements
 - same idea as beam in 2D
 Specify thickness

2.2 Material Models

Linear Elasticity (small strains)

$$\epsilon_{11} = \frac{\delta l}{l} = \frac{1}{E} \sigma_{11} \quad \epsilon_{22} = \epsilon_{33} = -\frac{\nu}{E} \sigma_{11}$$

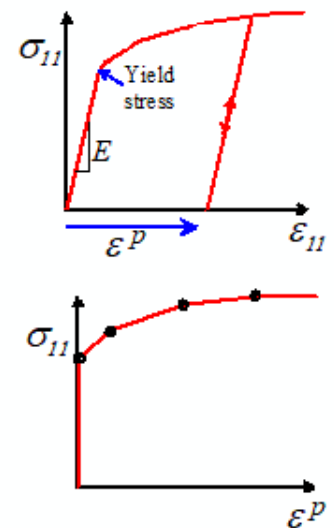
(Also anisotropic materials)



Elastic - plastic (strains can be large)

Material is nonlinear & irreversible

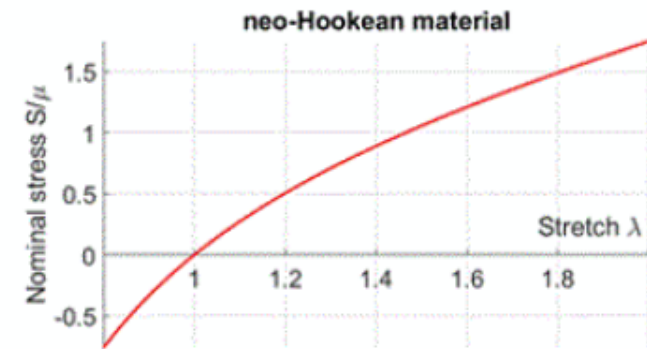
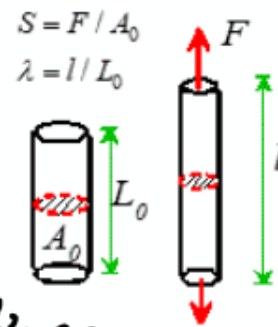
Specify yield stress - ν - plastic strain
in ABAQUS with table.



Hyperelasticity (large strain elasticity)

Rubber material; large deformations; nonlinear σ - ϵ curve; reversible

Need to use stress & strain measures for large deformations

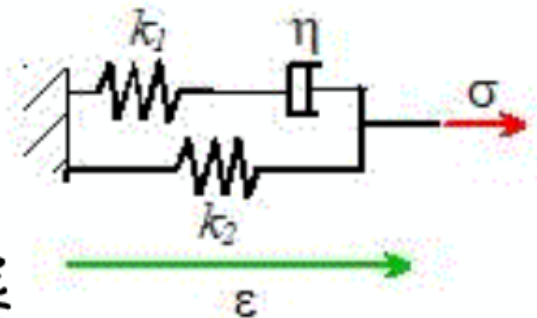


Usually approximated as near-incompressible

Viscoelasticity (rate dependent, can be linear or not)

Stress-strain behavior specified by ODE - for example shown

$$\frac{\eta}{k_2} \frac{d\sigma}{dt} + \sigma = k_2 \epsilon + \left(1 + \frac{k_1}{k_2}\right) \eta \frac{d\epsilon}{dt}$$



Or can specify variation of elastic moduli
with time