

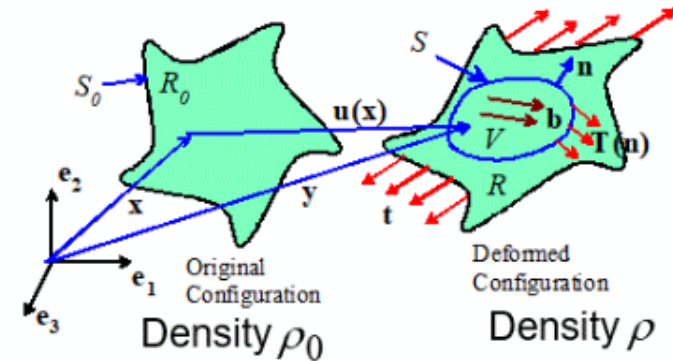
Review

Equations of motion for deformable solids

Assumptions

Internal forces characterized by Cauchy stress: σ_{ij}

Any internal volume in solid obeys Newton's laws



Linear momentum $\mathbf{F} = \frac{d}{dt} \{m\mathbf{v}\} \Rightarrow \nabla_{\mathbf{y}} \cdot \boldsymbol{\sigma} + \rho \mathbf{b} = \rho \frac{\partial \mathbf{v}}{\partial t}$

$$\frac{\partial \sigma_{ij}}{\partial y_i} + \rho b_j = \rho \frac{\partial v_j}{\partial t}$$

$$\frac{\partial \sigma_{11}}{\partial y_1} + \frac{\partial \sigma_{21}}{\partial y_2} + \frac{\partial \sigma_{31}}{\partial y_3} + \rho b_1 = \rho \frac{dv_1}{dt}$$

$$\frac{\partial \sigma_{12}}{\partial y_1} + \frac{\partial \sigma_{22}}{\partial y_2} + \frac{\partial \sigma_{32}}{\partial y_3} + \rho b_2 = \rho \frac{dv_2}{dt}$$

$$\frac{\partial \sigma_{13}}{\partial y_1} + \frac{\partial \sigma_{23}}{\partial y_2} + \frac{\partial \sigma_{33}}{\partial y_3} + \rho b_3 = \rho \frac{dv_3}{dt}$$

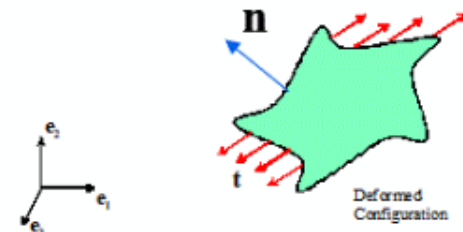
For small deformations OK to neglect difference between \mathbf{y} and \mathbf{x} $\frac{\partial \sigma_{ij}}{\partial x_i} + \rho_0 b_j \approx \rho_0 \frac{\partial v_j}{\partial t}$

Angular momentum $\mathbf{r} \times \mathbf{F} = \frac{d}{dt} \{ \mathbf{r} \times m\mathbf{v} \} \Rightarrow \boldsymbol{\sigma} = \boldsymbol{\sigma}^T$

$$\sigma_{ij} = \sigma_{ji}$$

Boundary conditions for stress at an exterior surface

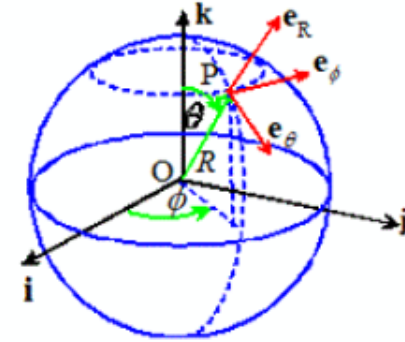
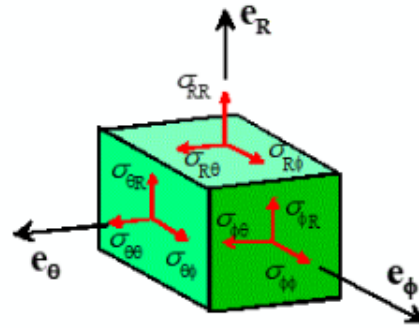
$$\mathbf{n}\boldsymbol{\sigma} = \mathbf{t} \quad n_i \sigma_{ij} = t_j$$



Other forms for equations of motion

Spherical-polar coordinates

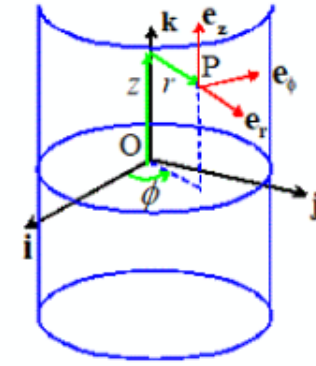
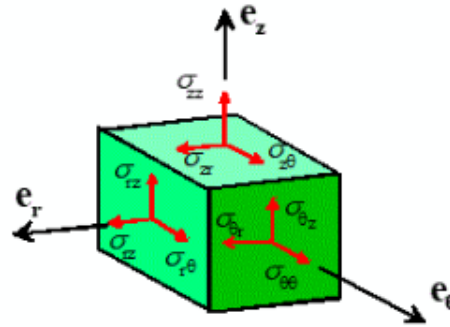
$$\sigma = \begin{bmatrix} \sigma_{RR} & \sigma_{R\theta} & \sigma_{R\phi} \\ \sigma_{\theta R} & \sigma_{\theta\theta} & \sigma_{\theta\phi} \\ \sigma_{\phi R} & \sigma_{\phi\theta} & \sigma_{\phi\phi} \end{bmatrix}$$



$$\nabla \cdot \sigma + \rho \mathbf{b} = \rho \frac{d\mathbf{v}}{dt} \equiv \begin{bmatrix} \frac{\partial \sigma_{RR}}{\partial R} + 2 \frac{\sigma_{RR}}{R} + \frac{1}{R} \frac{\partial \sigma_{\theta R}}{\partial \theta} + \cot \theta \frac{\sigma_{\theta R}}{R} + \frac{1}{R \sin \theta} \frac{\partial \sigma_{\phi R}}{\partial \phi} - \frac{1}{R} (\sigma_{\theta\theta} + \sigma_{\phi\phi}) \\ \frac{\partial \sigma_{R\theta}}{\partial R} + 2 \frac{\sigma_{R\theta}}{R} + \frac{1}{R} \frac{\partial \sigma_{\theta\theta}}{\partial \theta} + \cot \theta \frac{\sigma_{\theta\theta}}{R} + \frac{1}{R \sin \theta} \frac{\partial \sigma_{\phi\theta}}{\partial \phi} + \frac{\sigma_{\theta R}}{R} - \cot \theta \frac{\sigma_{\phi\phi}}{R} \\ \frac{\partial \sigma_{R\phi}}{\partial R} + 2 \frac{\sigma_{R\phi}}{R} + \frac{\sin \theta}{R} \frac{\partial \sigma_{\theta\phi}}{\partial \theta} + \cos \theta \frac{\sigma_{\theta\phi}}{R} + \frac{1}{R \sin \theta} \frac{\partial \sigma_{\phi\phi}}{\partial \phi} + \frac{1}{R} (\sigma_{\phi R} + \sigma_{\phi\theta}) \end{bmatrix} + \begin{bmatrix} \rho b_R \\ \rho b_\theta \\ \rho b_\phi \end{bmatrix} = \begin{bmatrix} \rho \frac{dv_R}{dt} \\ \rho \frac{dv_\theta}{dt} \\ \rho \frac{dv_\phi}{dt} \end{bmatrix}$$

Cylindrical-polar coordinates

$$\sigma = \begin{bmatrix} \sigma_{rr} & \sigma_{r\theta} & \sigma_{rz} \\ \sigma_{\theta r} & \sigma_{\theta\theta} & \sigma_{\theta z} \\ \sigma_{zr} & \sigma_{z\theta} & \sigma_{zz} \end{bmatrix}$$



$$\nabla \cdot \sigma + \rho \mathbf{b} = \rho \frac{d\mathbf{v}}{dt} = \begin{bmatrix} \frac{\partial \sigma_{rr}}{\partial r} + \frac{\sigma_{rr}}{r} + \frac{1}{r} \frac{\partial \sigma_{\theta r}}{\partial \theta} + \frac{\partial \sigma_{zr}}{\partial z} - \frac{\sigma_{\theta\theta}}{r} \\ \frac{1}{r} \frac{\partial \sigma_{\theta\theta}}{\partial \theta} + \frac{\partial \sigma_{r\theta}}{\partial r} + \frac{\sigma_{r\theta}}{r} + \frac{\sigma_{\theta r}}{r} + \frac{\partial \sigma_{z\theta}}{\partial z} \\ \frac{\partial \sigma_{zz}}{\partial z} + \frac{\partial \sigma_{rz}}{\partial r} + \frac{\sigma_{rz}}{r} + \frac{1}{r} \frac{\partial \sigma_{\theta z}}{\partial \theta} \end{bmatrix} + \begin{bmatrix} \rho b_r \\ \rho b_\theta \\ \rho b_z \end{bmatrix} = \begin{bmatrix} \rho \frac{dv_r}{dt} \\ \rho \frac{dv_\theta}{dt} \\ \rho \frac{dv_z}{dt} \end{bmatrix}$$

6.4 Power and work done by stresses

Rate of work done by stress per unit volume

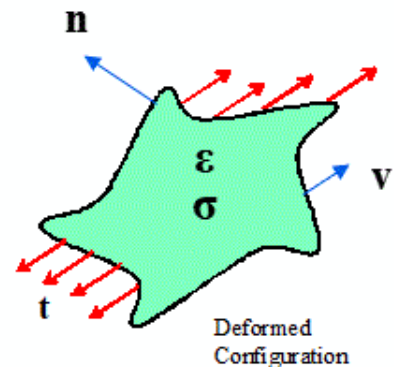
$$\frac{P}{V} = \underline{\sigma} : \frac{d\underline{\epsilon}}{dt} = \sigma_{ij} \frac{d\epsilon_{ij}}{dt} = \sigma_{11} \frac{d\epsilon_{11}}{dt} + \sigma_{12} \frac{d\epsilon_{12}}{dt} + \sigma_{13} \frac{d\epsilon_{13}}{dt} + \dots$$

Total work done per unit vol $\frac{W}{V} = \int_0^{\epsilon_{pe}} \underline{\sigma}_{ij}(\epsilon_{pe}) d\epsilon_{ij}$

Derivation: Consider solid under uniform stress & strain

Rate of work done by ext forces ($P = \underline{F} \cdot \underline{v}$)

$$P = \int_A \underline{t} \cdot \underline{v} dA$$



Recall $\underline{t} = \underline{n} \sigma$

For uniform strain $\underline{u} = \epsilon \underline{x} \Rightarrow \underline{v} = \frac{d\epsilon}{dt} \underline{x}$

Position on boundary

$$\Rightarrow P = \int_A n_i \sigma_{ij} \frac{d\epsilon_{jk}}{dt} x_k dA = \sigma_{ij} \frac{d\epsilon_{jk}}{dt} \int_A n_i x_k dA$$

Divergence Thm: $\int_A n_i x_k dA = \int_V \frac{\partial}{\partial x_i} x_k dV = \delta_{ik} V$

$$P = \sigma_{ij} \frac{d\epsilon_{jk}}{dt} \delta_{ik} = \sigma_{ij} \frac{d\epsilon_{ji}}{dt} = \sigma_{ij} \frac{d\epsilon_{ij}}{dt}$$

$$\sigma_{ij} = \sigma_{ji}$$

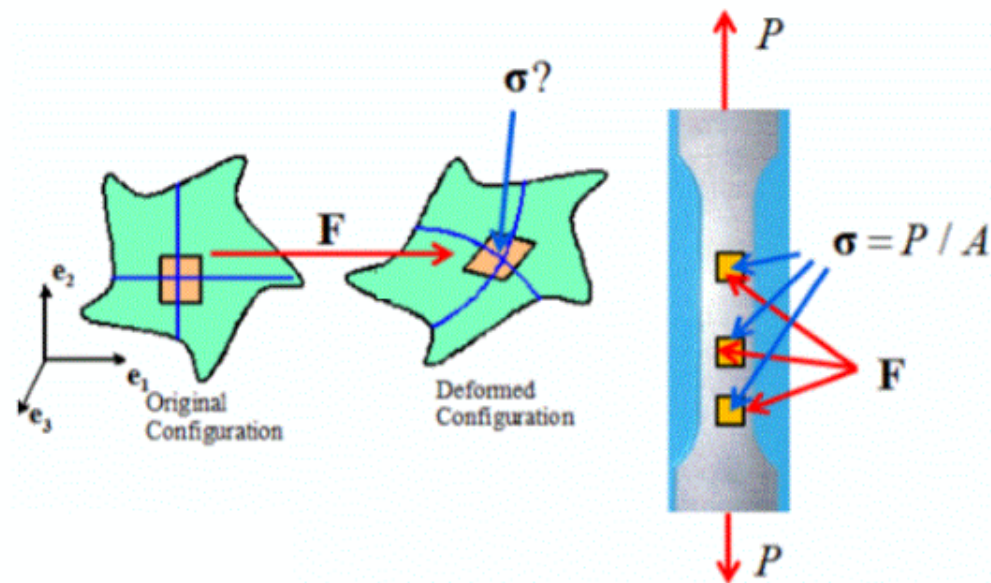
7 Stress-strain-temperature relations for small strain isotropic elastic materials ("linear elastic")

Background: We have (1) Strain-displacement $\underline{\epsilon} = \frac{1}{2} (\underline{\nabla} \underline{u} + \underline{\nabla} \underline{u}^T)$
 (2) EOM $\nabla \cdot \underline{\sigma} + \underline{\rho} \underline{b} = \underline{\rho} \underline{\partial} \underline{u} / \underline{\partial} t$
 (3) Boundary Conditions $\underline{t} = \underline{n} \underline{\sigma}$

Can't solve! Incomplete! Need to relate $\underline{\sigma}$ to $\underline{\epsilon}$

Engineering approach

- (1) Subject specimen to uniform strain $\Rightarrow \underline{\sigma}$ uniform
- (2) Measure force P , deduce $\underline{\sigma}$
- (3) Fit curve to results
- try to use physics

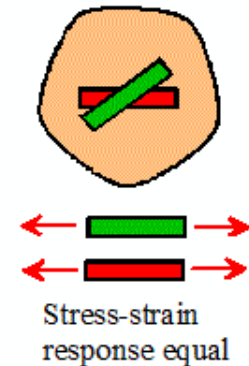
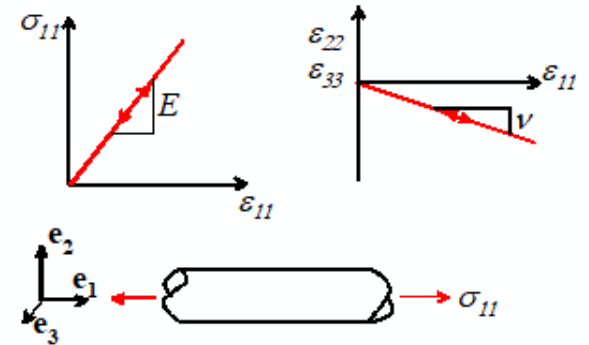


7.1 Stress-strain-temp relations

Experimental Observations

- (1) Stress is proportional to strain
- (2) Behavior is reversible
- (3) For most materials behavior is independent of orientation of specimen wrt material "isotropic"
- (4) Increasing temperature produces uniform volumetric strain proportional to temperature

Fit with curve



Matrix form of σ - ϵ relation (as in ABAQUS)

$$\underbrace{\begin{bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \epsilon_{33} \\ 2\epsilon_{12} \\ 2\epsilon_{13} \\ 2\epsilon_{23} \end{bmatrix}}_{\underline{\epsilon}} = \frac{1}{E} \underbrace{\begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix}}_{[S] : \text{"Compliance Tensor"}} \underbrace{\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{12} \\ \sigma_{13} \\ \sigma_{23} \end{bmatrix}}_{\underline{\sigma}} + \Delta T \underbrace{\begin{bmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}}_{\underline{\alpha} \text{ Thermal exp. tensor}}$$

E : Young's Modulus ν : Poisson's ratio
 α : thermal expansion coeft

$$\underline{\epsilon} = [S] \underline{\sigma} + \underline{\alpha} \Delta T$$

$\underline{\epsilon}^e$: "elastic" strain

Inverse: $\underline{\sigma} = \underbrace{[S]^{-1}}_{[C] : \text{Stiffness tensor}} (\underline{\epsilon} - \underline{\alpha} \Delta T)$

Index Notation

$$\varepsilon_{ij} = S_{ijke} \bar{\sigma}_{ke} + \alpha \Delta T \delta_{ij}$$

$$\bar{\sigma}_{ij} = C_{ijke} (\varepsilon_{ke} - \alpha \Delta T \delta_{ij})$$

or
$$\varepsilon_{ij} = \frac{1+\nu}{E} \bar{\sigma}_{ij} - \frac{\nu}{E} \bar{\sigma}_{kk} \delta_{ij} + \alpha \Delta T \delta_{ij}$$

$$\bar{\sigma}_{ij} = \frac{E}{1+\nu} \left\{ \varepsilon_{ij} + \frac{\nu}{1-2\nu} \varepsilon_{kk} \delta_{ij} \right\} - \frac{E\alpha\Delta T}{1-2\nu} \delta_{ij}$$

Example: Uniaxial Tension

$$\bar{\sigma}_{11} = \bar{\sigma}_0$$

All others zero

σ - ε relations predict

$$\varepsilon_{11} = \frac{\bar{\sigma}_0}{E}; \varepsilon_{33} = \varepsilon_{22} = -\frac{\nu}{E} \bar{\sigma}_0$$

All others zero

Other elastic constants

Bulk modulus $\kappa = \frac{E}{3(1-2\nu)}$

Significance $\bar{\sigma}_h = \kappa \frac{\Delta V}{V}$ $\bar{\sigma}_h = \frac{1}{3} \text{trace}(\sigma)$
 $\frac{\Delta V}{V} = \text{trace}(\varepsilon)$

Shear modulus $\mu = \frac{E}{2(1+\nu)}$

Significance $\bar{\sigma}_{12} = \mu 2\varepsilon_{12}$ $\bar{\sigma}_{13} = \mu 2\varepsilon_{13}$ etc

Lamé' modulus $\lambda = \frac{\nu E}{(1+\nu)(1-2\nu)}$

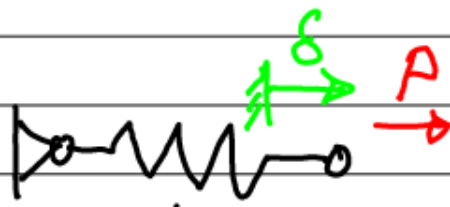
Significance

$$\bar{\sigma}_{ij} = 2\mu \varepsilon_{ij} + \lambda \varepsilon_{kk} \delta_{ij} \rightarrow (3\lambda + 2\mu) \alpha \Delta \delta_{ij}$$

Relations between elastic constants

| | LAME MODULUS λ | SHEAR MODULUS μ | YOUNG'S MODULUS E | POISSON'S RATIO ν | BULK MODULUS K |
|----------------|-------------------------------------|-----------------------------------|--|------------------------------------|-------------------------------------|
| λ, μ | | | $\frac{\mu(3\lambda + 2\mu)}{\lambda + \mu}$ | $\frac{\lambda}{2(\lambda + \mu)}$ | $\frac{3\lambda + 2\mu}{3}$ |
| λ, E | | Irrational | | Irrational | Irrational |
| λ, ν | | $\frac{\lambda(1 - 2\nu)}{2\nu}$ | $\frac{\lambda(1 + \nu)(1 - 2\nu)}{\nu}$ | | $\frac{\lambda(1 + \nu)}{3\nu}$ |
| λ, K | | $\frac{3(K - \lambda)}{2}$ | $\frac{9K(K - \lambda)}{3K - \lambda}$ | $\frac{\lambda}{3K - \lambda}$ | |
| μ, E | $\frac{\mu(2\mu - E)}{E - 3\mu}$ | | | $\frac{E - 2\mu}{2\mu}$ | $\frac{\mu E}{3(3\mu - E)}$ |
| μ, ν | $\frac{2\mu\nu}{1 - 2\nu}$ | | $2\mu(1 + \nu)$ | | $\frac{2\mu(1 + \nu)}{3(1 - 2\nu)}$ |
| μ, K | $\frac{3K - 2\mu}{3}$ | | $\frac{9K\mu}{3K + \mu}$ | $\frac{3K - 2\mu}{2(3K + \mu)}$ | |
| E, ν | $\frac{\nu E}{(1 + \nu)(1 - 2\nu)}$ | $\frac{E}{2(1 + \nu)}$ | | | $\frac{E}{3(1 - 2\nu)}$ |
| E, K | $\frac{3K(3K - E)}{9K - E}$ | $\frac{3EK}{9K - E}$ | | $\frac{3K - E}{6K}$ | |
| ν, K | $\frac{3K\nu}{(1 + \nu)}$ | $\frac{3K(1 - 2\nu)}{2(1 + \nu)}$ | $3K(1 - 2\nu)$ | | |

7.2 Strain energy density

Idea: energy stored in elastic deformation 

Energy per unit volume

$$U = \frac{1}{2} k \delta^2$$

$$U = \frac{1}{2} \underline{\sigma} \cdot \underline{\varepsilon}^e = \frac{1}{2} \sigma_{ij} \varepsilon_{ij}^e = \frac{1}{2} (\sigma_{11} \varepsilon_{11}^e + \sigma_{22} \varepsilon_{22}^e + \sigma_{33} \varepsilon_{33}^e + \sigma_{21} \varepsilon_{21}^e + \text{etc})$$

Here $\underline{\sigma} = [\sigma_{11}, \sigma_{22}, \sigma_{33}, \sigma_{12}, \sigma_{13}, \sigma_{23}]$

$$\underline{\varepsilon}^e = [\varepsilon_{11} - \alpha \Delta T, \varepsilon_{22} - \alpha \Delta T, \varepsilon_{33} - \alpha \Delta T, 2\varepsilon_{12}, 2\varepsilon_{13}, 2\varepsilon_{23}]$$

Also useful $U = \frac{1}{2} ([C] \underline{\varepsilon}^e) \cdot \underline{\varepsilon}^e = \frac{1}{2} C_{ijke} \varepsilon_{ij}^e \varepsilon_{ke}^e$

$$= \frac{1}{2} ([S] \underline{\sigma}) \cdot \underline{\sigma} = \frac{1}{2} S_{ijke} \sigma_{ij} \sigma_{ke}$$

Derivation :

$$\begin{aligned}
 U &= \frac{W}{V} = \int_0^{\epsilon} \sigma_{ij}(\epsilon^e) d\epsilon_{ij}^e \\
 &= \int_0^{\epsilon_{ij}^e} C_{ijke} \epsilon_{ke}^e d\epsilon_{ij}^e \\
 &= \frac{1}{2} C_{ijke} \epsilon_{ke}^e \epsilon_{ij}^e
 \end{aligned}$$

U will be needed to calculate potential energy of a solid to derive FEA equations

7.3 Simplified stress & strain states

Plane stress : thin sheet loaded in $\{e_1, e_2\}$ plane

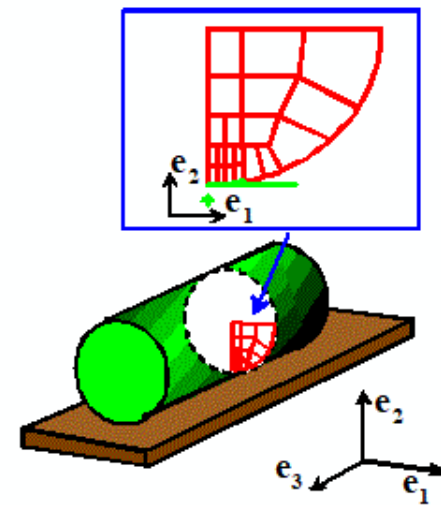
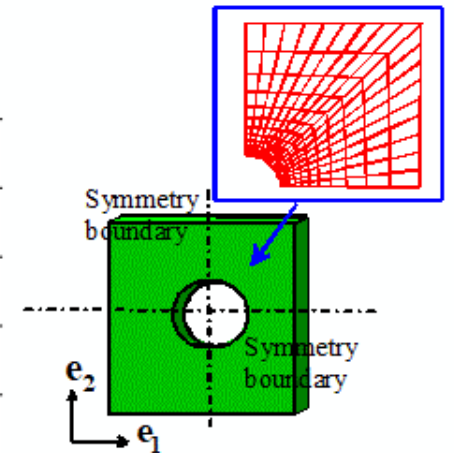
Traction free surfaces $[0 \ 0 \ 1] \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$

$$\Rightarrow \sigma_{33} = \sigma_{13} = \sigma_{23} = 0$$

Plane Strain Long cylinder with no axial deformation

$$u_3 = 0 \quad ; \quad u_1, u_2 \text{ depend only on } x_1, x_2$$

$$\epsilon_{33} = \epsilon_{13} = \epsilon_{23} = 0$$



Useful elasticity formulas

Matrix form for stress-strain law (3D)

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ 2\varepsilon_{23} \\ 2\varepsilon_{13} \\ 2\varepsilon_{12} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & -\nu & 0 & 0 & 0 \\ -\nu & 1 & -\nu & 0 & 0 & 0 \\ -\nu & -\nu & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 2(1+\nu) & 0 & 0 \\ 0 & 0 & 0 & 0 & 2(1+\nu) & 0 \\ 0 & 0 & 0 & 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{bmatrix} + \alpha \Delta T \begin{bmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix} \quad \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{13} \\ \sigma_{12} \end{bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & \nu & 0 & 0 & 0 \\ \nu & 1-\nu & \nu & 0 & 0 & 0 \\ \nu & \nu & 1-\nu & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{(1-2\nu)}{2} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{(1-2\nu)}{2} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{(1-2\nu)}{2} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ 2\varepsilon_{23} \\ 2\varepsilon_{13} \\ 2\varepsilon_{12} \end{bmatrix} - \frac{E\alpha\Delta T}{1-2\nu} \begin{bmatrix} 1 \\ 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

For plane strain $\varepsilon_{33} = \varepsilon_{23} = \varepsilon_{13} = 0$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} = \frac{(1+\nu)}{E} \begin{bmatrix} 1-\nu & -\nu & 0 \\ -\nu & 1-\nu & 0 \\ 0 & 0 & 2 \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} + (1+\nu)\alpha\Delta T \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix} \quad \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} - \frac{E\alpha\Delta T}{1-2\nu} \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix}$$

$$\sigma_{33} = \frac{E\nu(\varepsilon_{11} + \varepsilon_{22})}{(1-2\nu)(1+\nu)} + \frac{E\alpha\Delta T}{1-2\nu}, \quad \sigma_{13} = \sigma_{23} = 0$$

For plane stress $\sigma_{33} = \sigma_{23} = \sigma_{13} = 0$

$$\begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & 0 \\ -\nu & 1 & 0 \\ 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} + \alpha\Delta T \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix} \quad \begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu)/2 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} - \frac{E\alpha\Delta T}{(1-\nu)} \begin{bmatrix} 1 \\ 1 \\ 0 \end{bmatrix}$$

$$\varepsilon_{33} = -\frac{\nu}{E}(\sigma_{11} + \sigma_{22}) + \alpha\Delta T$$