

9.2 Hyperelasticity - FEA for finite strains

Governing equations

Def gradient $\frac{\partial U_i}{\partial x_j} = S_{ij} + \frac{\partial U_i}{\partial x_j} = F_{ij}$

- Jacobian $J = \det(F)$
- Cauchy - Green tensors

$$B_{ij} = F_{ik} F_{jk}$$

$$C_{ij} = F_{ki} F_{kj}$$

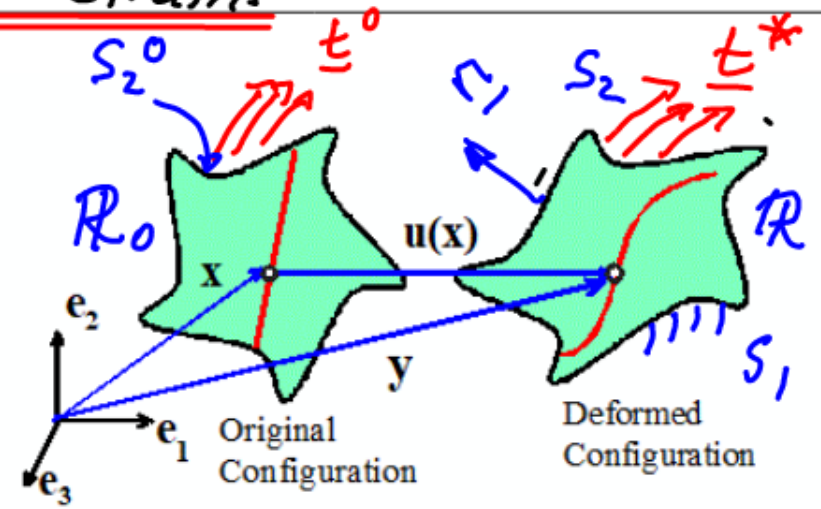
- Stresses :

σ_{ij} - Cauchy "true" stress

τ_{ij} - Kirchhoff stress $\tau_{ij} = J \sigma_{ij}$

$S_{ij} = F_{ik} J^{-1} \sigma_{kj}$ - First Piola-Kirchhoff / Nominal

$\Sigma_{ij} = F_{ik}^{-1} J^{-1} \sigma_{kl} F_{jl}^{-1}$ 2nd P-K "material"



• Equilibrium $\frac{\partial \sigma_{ji}}{\partial y_j} = 0$ $\frac{\partial \xi_{ji}}{\partial x_j} = 0$

• Boundary conditions $n_i \sigma_{ij} = t_j^*$ on S_2
 $u_i = u_i^*$ on S_1

• Stress - Strain relation (Neo-Hookean solid)

$$\sigma_{ij} = \frac{1}{J^{5/3}} \mu \left\{ B_{ij} - \frac{B_{kk}}{3} \delta_{ij} \right\} + k(J-1) \delta_{ij}$$

μ - shear modulus $k \gg \mu$ Bulk modulus

(Simple rubber model)

Weak form of equilibrium (Use 1st version)

$$\int_{\mathcal{R}} \frac{\partial \bar{\sigma}_{ji}}{\partial y_j} \eta_i dV = 0 \quad \forall \text{ admiss } \eta_i$$

$$\Rightarrow \int_{\mathcal{R}} \left\{ \frac{\partial}{\partial y_j} (\bar{\sigma}_{ji} \eta_i) - \bar{\sigma}_{ji} \frac{\partial \eta_i}{\partial y_j} \right\} dV = 0$$

$$\Rightarrow \int_{\mathcal{R}} \bar{\sigma}_{ji} \frac{\partial \eta_i}{\partial y_j} dV - \int_{S_2} t_i^* \eta_i dA = 0 \quad \forall \eta_i$$

Map back to ref config

$$\int_{\mathcal{R}_0} \underbrace{J \bar{\sigma}_{ji}}_{= \bar{\sigma}_{ji}^0} \frac{\partial \eta_i}{\partial y_j} dV_0 - \int_{S_2^0} t_i^0 \eta_i dA_0 = 0 \quad \forall \eta_i$$

FE approximation

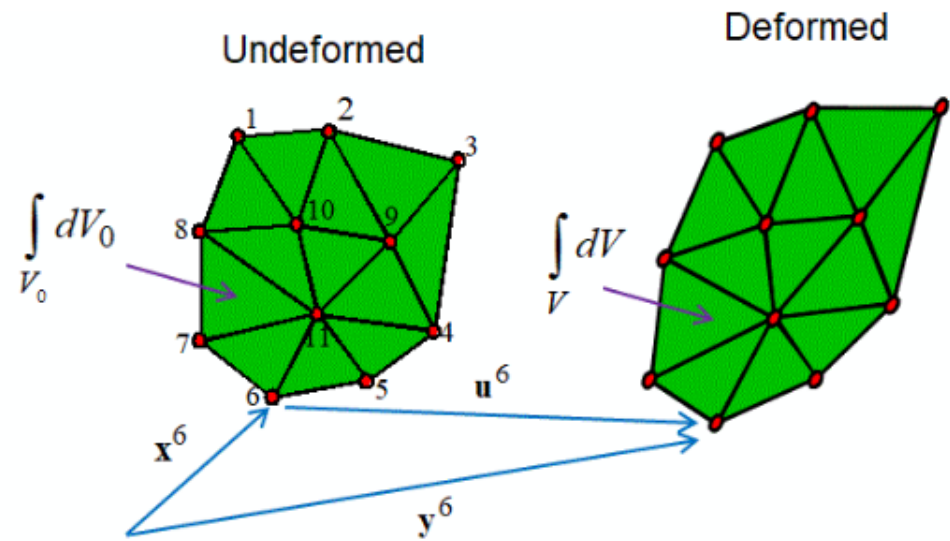
$$u_i = N^a(\underline{x}) u_i^a \quad \eta_i = N^a \eta_i^a$$

Discrete equilibrium eq

$$R_i^a = f_i^a$$

$$R_i^a = \int_{\mathcal{V}_0} \tau_{ji} \frac{\partial N^a}{\partial y_j} dV_0$$

$$f_i^a = \int_{S_2^0} t_i^0 N^a dA_0$$

Computing R_i^a

$$(1) \text{ Find } \frac{\partial N^a}{\partial \xi_i} \text{ \& Find } \frac{\partial x_i}{\partial \xi_j} = \sum_a \frac{\partial N^a}{\partial \xi_j} x_i^a$$

$$(2) \frac{\partial N^a}{\partial x_j} = \left[\frac{\partial N^a}{\partial \xi_k} \right] \left[\frac{\partial \xi_k}{\partial x_j} \right]^{-1}$$

$$(3) \frac{\partial N^a}{\partial y_j} = \frac{\partial N^a}{\partial x_k} \frac{\partial x_k}{\partial y_j} = \frac{\partial N^a}{\partial x_k} F_{kj}$$

Coding : (1) Find $\partial N^a / \partial x_j$ (steps 1, 2)

$$(2) F_{ij} = \frac{\partial N^a}{\partial x_j} U_i^a + \delta_{ij}$$

$$(3) F_{ij}^{-1}$$

$$(4) \frac{\partial N^a}{\partial y_j} = \frac{\partial N^a}{\partial x_k} F_{kj}^{-1}$$

Express R_i^a as vector (usual convention)

$$\underline{R} = [R_1^1 \ R_2^1 \ R_3^1 \ R_1^2 \ \dots]$$

$$\underline{R}^{el} = \int_{\Omega_0} [B]^T \underline{\tau} dV_0 \quad \underline{\tau} = [\tau_{11} \ \tau_{22} \ \tau_{33} \ \tau_{12} \ \dots]$$

$[B]$ is usual $[B]$ but $\partial N^a / \partial x_j$ is replaced by $\partial N^a / \partial y_j$

For example 3D $[B]$ matrix has form shown

Vol integrals are computed using same Gauss quadrature we used for small strains

$$\underline{R}^{el} = \sum_{\text{int pts } i} [B]^T \underline{\epsilon} w_i$$

$$[B] = \begin{bmatrix} \frac{\partial N^1}{\partial y_1} & 0 & 0 & \frac{\partial N^2}{\partial y_1} & 0 & 0 \\ 0 & \frac{\partial N^1}{\partial y_2} & 0 & 0 & \frac{\partial N^2}{\partial y_2} & 0 \\ 0 & 0 & \frac{\partial N^1}{\partial y_3} & 0 & 0 & \frac{\partial N^2}{\partial y_3} \\ \frac{\partial N^1}{\partial y_2} & \frac{\partial N^1}{\partial y_1} & 0 & \frac{\partial N^2}{\partial y_2} & \frac{\partial N^2}{\partial y_1} & 0 \\ \frac{\partial N^1}{\partial y_3} & 0 & \frac{\partial N^1}{\partial y_1} & & & \\ 0 & \frac{\partial N^1}{\partial y_3} & \frac{\partial N^1}{\partial y_2} & & & \end{bmatrix}$$

We now have nonlinear eqs $R_i^a [U_k^b] = f_i^a$

Solve nonlinear eqs with Newton-Raphson

- Guess $\underline{u} = \underline{w}$ (eg $\underline{w} = 0$ or sol from previous step)

- Correct \underline{w} : $[\underline{K}] d\underline{w} = \underline{f} - \underline{R}[\underline{w}]$

$$[\underline{K}] = \frac{\partial \underline{R}}{\partial \underline{u}} \quad (\text{consistent tangent})$$

$$\underline{w} \rightarrow \underline{w} + d\underline{w}$$

- Check convergence, repeat as needed

$$\text{We need } \frac{d\underline{R}}{d\underline{u}} \equiv \frac{\partial R_i^a}{\partial u_k^b} = \frac{\partial}{\partial u_k^b} \left\{ \int_{R_0} \frac{\partial N^a}{\partial y_j} \tau_{ji} dV_0 \right\}$$

$$\equiv \underline{K}_{aibk}$$

Chain rule

$$K_{aibk} = \int_{\mathcal{R}_0} \frac{\partial N^a}{\partial y_i} \frac{\partial \tau_{ji}}{\partial B_{pq}} \frac{\partial B_{pq}}{\partial F_{rs}} \frac{\partial F_{rs}}{\partial U_k^b} dV_0 + \int_{\mathcal{R}} \frac{\partial}{\partial U_k^b} \left\{ \frac{\partial N^a}{\partial y_j} \right\} \tau_{ji} dV_0$$

Similar to small strains

Geometric stiffness

Derivatives

$$F_{rs} = \delta_{rs} + \frac{\partial N^c}{\partial x_s} U_r^c$$

$$\Rightarrow \frac{\partial F_{rs}}{\partial U_k^b} = \frac{\partial N^b}{\partial x_s} \delta_{kr} = \frac{\partial N^b}{\partial y_l} F_{ls} \delta_{kr}$$

$$B_{pq} = F_{pn} F_{qn} \Rightarrow \frac{\partial B_{pq}}{\partial F_{rs}} = \delta_{pr} \delta_{ns} F_{qn} + F_{pn} \delta_{qr} \delta_{ns} \\ = \delta_{pr} F_{qs} + F_{ps} \delta_{qr}$$

$$\frac{\partial}{\partial U_k^b} \left\{ \frac{\partial N^a}{\partial y_j} \right\} = \left\{ \frac{\partial N^a}{\partial x_s} F_{sj}^{-1} \right\} \quad (1)$$

Note $FF^{-1} = I \Rightarrow \frac{\partial F}{\partial \lambda} F^{-1} + F \frac{\partial F^{-1}}{\partial \lambda} = 0$

$$\Rightarrow \frac{\partial F^{-1}}{\partial \lambda} = -F^{-1} \frac{\partial F}{\partial \lambda} F^{-1}$$

$$\Rightarrow (1) = - \frac{\partial N^a}{\partial x_s} F_{sp}^{-1} \frac{\partial F_{pr}}{\partial U_k^b} F_{rj}^{-1} \quad \delta_{jk}$$

$$= - \underbrace{\frac{\partial N^a}{\partial x_s}}_{\frac{\partial N^a}{\partial y_p}} F_{sp}^{-1} \frac{\partial N^b}{\partial y_e} F_{er} \underbrace{\delta_{rp}}_{\delta_{jk}} F_{rj}^{-1} = - \frac{\partial N^a}{\partial y_k} \frac{\partial N^b}{\partial y_j}$$

Combine terms so far

$$K_{aibk}^{el} = \int_{V_0} \frac{\partial N^a}{\partial y_j} \frac{\partial \tau_{ji}}{\partial B_{pq}} (\delta_{pk} B_{ql} + \delta_{qk} B_{pl}) \frac{\partial N^b}{\partial y_l} dV_0 - \int_{V_0} \frac{\partial N^a}{\partial y_k} \tau_{ji} [F_{kl}] \frac{\partial N^b}{\partial y_j} dV_0$$

Finally we need material tangent stiffnesses $\frac{\partial \tau_{ji}}{\partial B_{pq}}$

$$\tau_{ji} = \frac{\mu}{J^{2/3}} \left\{ B_{ij} - B_{kk} \frac{\delta_{ij}}{3} \right\} + K J (J-1) \delta_{ij}$$

Note $J \approx \sqrt{\det(B)}$ $\frac{\partial \det(A)}{\partial A} = \det(A) A^{-T}$

$$\frac{\partial \tau_{ij}}{\partial B_{pq}} = \frac{M}{J^{2/3}} \left\{ \frac{\delta_{ip} \delta_{jq} + \delta_{iq} \delta_{jp}}{2} - \frac{\delta_{pq} \delta_{ij}}{3} \right\}$$

$$- \frac{M}{J^{2/3}} \left\{ B_{ij} - \frac{B_{kk} \delta_{ij}}{3} \right\} B_{pq}^{-1}$$

$$+ KJ \left(J - \frac{1}{2} \right) \delta_{ij} B_{pq}^{-1}$$

↖ Symmetrize to preserve symmetry
in ij & p,q

See notes for L14 for matrix forms for $[k]$
and sample UEL code