

# EN 4 HW #6 SOLUTIONS

1

7 pts. 1. a) The cantilever acts like a spring with spring constant  $k = 3EI/L^3$ . The frequency of vibration for a mass  $m_{\text{eff}}$  attached to this spring is then

1 pt. 
$$\omega_n = \sqrt{\frac{k}{m_{\text{eff}}}} = \sqrt{\frac{3EI}{L^3 m_{\text{eff}}}}$$

We choose  $m_{\text{eff}}$  so this frequency matches the true frequency  $\omega_n = 1.03 \sqrt{Ewt^3/l^3 m}$  where  $l = L$ . So

2 pts. 
$$(1.03)^2 \frac{3EI}{L^3 m_{\text{eff}}} = \frac{Ewt^3}{L^3 m} \Rightarrow \frac{m_{\text{eff}}}{m} = \frac{(1.03)^2 3I}{wt^3}$$

For a rectangular beam,  $I = \frac{wt^3}{12}$  so  $\frac{m_{\text{eff}}}{m} = \frac{(1.03)^2}{12} = 0.235$

b) Upon adding the E-coli, of mass  $m_{\text{ec}}$ , the natural frequency becomes

2 pts. 
$$\omega'_n = \sqrt{\frac{3EI}{L^3(m_{\text{eff}} + m_{\text{ec}})}} = \sqrt{\frac{3EI}{L^3 m_{\text{eff}}}} \frac{1}{\sqrt{1 + m_{\text{ec}}/m_{\text{eff}}}}$$

or 
$$\omega'_n = \omega_n \frac{1}{(1 + m_{\text{ec}}/m_{\text{eff}})^{1/2}}$$

The change in frequency is then

$$\Delta\omega_n = \omega'_n - \omega_n = \omega_n \left( \frac{1}{(1 + m_{\text{ec}}/m_{\text{eff}})^{1/2}} - 1 \right)$$

If  $m_{\text{ec}}/m_{\text{eff}} \ll 1$ , then  $(1 + m_{\text{ec}}/m_{\text{eff}})^{-1/2} \sim 1 - \frac{1}{2} \frac{m_{\text{ec}}}{m_{\text{eff}}}$

so

$$\Delta\omega_n = \omega_n \left( -\frac{1}{2} \frac{m_{\text{ec}}}{m_{\text{eff}}} \right) = \omega_n \left( -\frac{1}{2} \frac{m}{m_{\text{eff}}} \frac{m_{\text{ec}}}{m} \right) = 2.12 \omega_n \left( \frac{m_{\text{ec}}}{m} \right)$$

in terms of the original natural frequency and the true mass  $m$ .

$$\rightarrow m = 2.72 \times 10^{-13} \text{ kg} \quad (2)$$

$\omega_n$  for the beam shown is, using  $m = (\omega L t) \rho$ ,

$$\omega_n = 1.03 \sqrt{\frac{E \omega t^3}{L^3 \omega L t \rho}} = 1.03 \sqrt{\frac{E t^2}{L^4 \rho}} = 1.03 \sqrt{\frac{110 \times 10^9 \text{ N/m}^2 (325 \times 10^{-9} \text{ m})^2}{(25 \times 10^{-6} \text{ m})^4 3.4 \times 10^3 \text{ kg}/(10^{-2} \text{ m})^3}}$$

or

$$\omega_n = 2.9 \times 10^6 \frac{\text{rad}}{\text{s}} \Rightarrow f_n = 4.63 \times 10^5 \text{ Hz} = 463 \text{ kHz}$$

From the middle figure showing data vs. # of E-coli cells, we can estimate  $\Delta f_n = 3 \text{ kHz}$ . So,

$$\Delta f_n = (3 \text{ kHz}) = (463 \text{ kHz}) (2.12) (M_{ec}/m)$$

2 pts.

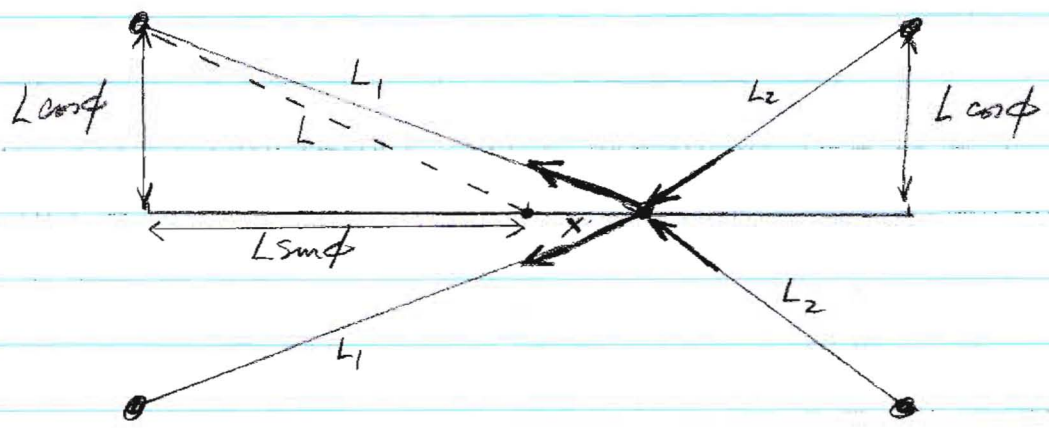
$$\Rightarrow M_{ec}/m = 3.05 \times 10^{-3} \Rightarrow M_{ec} = 3.05 \times 10^{-16} \text{ m}$$

$$M_{ec} = 8.31 \times 10^{-16} \text{ kg}$$

12 pts. 2. At equilibrium, the forces due to the four springs all cancel. If the mass is displaced by  $x$  then the spring forces change by  $k(\text{change in spring length})$  and the angle changes slightly as well. The initial length does matter, so we take  $L$  to be the unstretched length. To get  $\omega_n$ , we need to draw a F.B.D. and determine the EOM.

F.B.D. (exaggerated)

3 pts.



The  $y$  forces all balance. Summing the forces in the  $x$  direction we obtain

3 pts.

$$\sum F_x = \underbrace{-2k(L_1 - L)}_{\text{magnitude}} \underbrace{\frac{L \sin \phi + x}{L_1}}_{x \text{ component}} + 2k(L_2 - L) \left( \frac{L \sin \phi - x}{L_2} \right)$$

By geometry,

$$L_1 = \left( (L \sin \phi + x)^2 + (L \cos \phi)^2 \right)^{1/2} = \left( L^2 + 2L \sin \phi x + x^2 \right)^{1/2} \\ = L \left( 1 + 2 \sin \phi \frac{x}{L} + \frac{x^2}{L^2} \right)^{1/2} \\ \approx L \left( 1 + \sin \phi \frac{x}{L} \right) \quad \frac{x}{L} \ll 1 \\ = L + x \sin \phi$$

and

$$L_2 = \left( (L \sin \phi - x)^2 + (L \cos \phi)^2 \right)^{1/2} = \left( L^2 - 2L \sin \phi x + x^2 \right)^{1/2} \\ = L \left( 1 - 2 \sin \phi \frac{x}{L} + \frac{x^2}{L^2} \right)^{1/2} \\ \approx L \left( 1 - \sin \phi \frac{x}{L} \right) = L - x \sin \phi$$

2 pts So,  $L_1 - L \approx x \sin \phi$ ,  $L_2 - L \approx -x \sin \phi$   
 Plugging in:

$$\begin{aligned} \Sigma F_x &= -2k \sin \phi x \left( \frac{L \sin \phi + x}{L + x \sin \phi} \right) - 2k \sin \phi x \left( \frac{L \sin \phi - x}{L - x \sin \phi} \right) \\ &= \frac{4(\sin \phi + x/L)}{4(1 + \sin \phi x/L)} \quad = \quad \frac{4(\sin \phi - x)}{4(1 - \sin \phi x/L)} \\ &\approx \sin \phi \quad x/L \ll 1 \quad \approx \sin \phi \quad x/L \ll 1 \end{aligned}$$

→ terms  $\sim x/L$  will be multiplied by  $x$  in the force and so give terms  $\sim x^2$  and all all small.

2 pts. So

$$\Sigma F_x \approx -2k \sin \phi x (\sin \phi) - 2k \sin \phi x (\sin \phi)$$

$$\Sigma F_x = -4k \sin^2 \phi x$$

Makes sense. At  $\phi=90$ , we get  $4k$  - all four springs act together.  
 At  $\phi=0$ , we get 0 - springs don't contribute any force at order  $x/L$ .  
 (see problem 3 if  $T=0$ , also!)

$$\Rightarrow \underline{F} = \underline{ma} \Rightarrow m \frac{d^2 x}{dt^2} = -4k \sin^2 \phi x$$

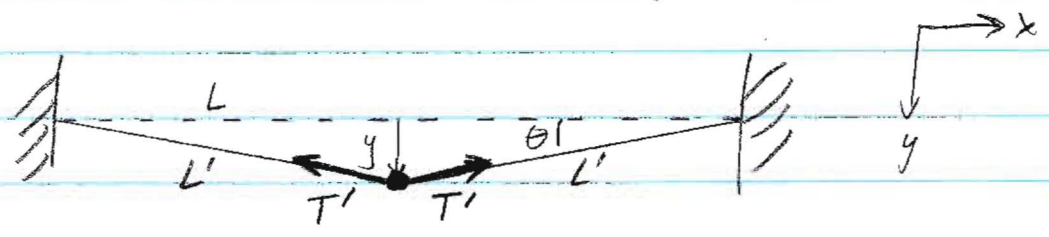
$$\Rightarrow \frac{d^2 x}{dt^2} + \frac{4k \sin^2 \phi}{m} x = 0$$

2 pts

$$\Rightarrow \omega_n = \sqrt{\frac{4k \sin^2 \phi}{m}}$$

10 pts 3. As in Prob. 2, draw F.B.D and get EOM

3 pts



Tension  $T'$  is unknown at the moment. When  $y=0$ , tension is  $T$ . For  $y \neq 0$ , the tension  $T'$  will change due to the change in length of the wire. However, that change in length is quite small, so that  $T' = T$  is accurate for  $y/L \ll 1$ .

Let's see this explicitly; from geometry we have

$$L' = (L^2 + y^2)^{1/2} = L(1 + y^2/L^2)^{1/2} \sim L(1 + \frac{1}{2} y^2/L^2)$$

2 pts.

so

$$\Delta L = L' - L = \frac{1}{2} y^2/L$$

The tension, assuming the wires act like linear springs is then

$$T' = T + k \Delta L \quad \text{extra tension due to extra stretch}$$

So,

2 pts.

$$\sum F_y = -2T' \sin \theta = -2T' \left( \frac{y}{L} \right)$$

$$= -2(T + k \Delta L) \frac{y}{L + \Delta L}$$

$$= -2 \left( T + \frac{1}{2} k y^2/L \right) \frac{y}{L(1 + \frac{1}{2} y^2/L^2)}$$

Since there is a factor of  $y$  just from  $\sin \theta$ , all other terms with  $y^2$  are higher order and can be neglected for small  $y$ .

Thus,

2 pts.

$$\sum F_y \approx -2Ty/L \quad k \text{ does not appear!}$$

Then

$$\tilde{F} = m \tilde{a} \Rightarrow m \frac{d^2 y}{dt^2} + \frac{2T}{L} y = 0 \Rightarrow \frac{d^2 y}{dt^2} + \frac{2T}{mL} y = 0$$

1 pt.

and so 
$$\omega_n = \sqrt{\frac{2T}{mL}}$$

For a frequency of  $f = 110$  Hz, with  $T = 750$  N and  $L = 1$  m, we have

$$2\pi(110) \frac{1}{5} = \sqrt{\frac{2(750)}{m(1m)}}$$

$$\Rightarrow m = \frac{1500 \text{ kg m/s}^2}{(2\pi(110))^2 \frac{1}{5^2} 1m} = .0031 \text{ kg} = \underline{\underline{3.1 \text{ grams}}}$$

6 pts 4.  $\tau_d$ : The time between peaks in the amplitude is 0.1 s  
 2 pts so  $\underline{\underline{\tau_d = 0.1 \text{ s}}}$

5: The peak heights at two successive peaks give the "log decrement"  $\delta$ , which is related to  $\zeta$  as  
 2 pts 
$$\zeta = \frac{\delta}{\sqrt{\delta^2 + (2\pi)^2}}$$

Here,  $\delta = \ln\left(\frac{.097}{.027}\right) = 1.28$  so  $\underline{\underline{\zeta = 0.2}}$

2 pts.  $\omega_n = \tau_d = \frac{2\pi}{\omega_d}$ ;  $\omega_d = \sqrt{1 - \zeta^2} \omega_n$  so

$$\omega_n = \frac{2\pi}{\tau_d \sqrt{1 - \zeta^2}} = \underline{\underline{64.1 \frac{\text{rad}}{\text{s}}}}$$