

HW9_Sol

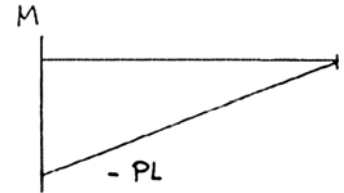
SOLUTION 11.63

Bending moment:

$$M = -Pv$$

Over AB:

$$\begin{aligned} U_{AB} &= \int_{L/2}^L \frac{M^2}{4EI} dv = \frac{P^2}{4EI} \int_{L/2}^L v^2 dv \\ &= \frac{P^2}{12EI} \left[L^3 - \left(\frac{L}{2} \right)^3 \right] = \frac{7}{96} \frac{P^2 L^3}{EI} \end{aligned}$$



Over BC:

$$\begin{aligned} U_{BC} &= \int_0^{L/2} \frac{M^2}{2EI} dv = \frac{P^2}{2EI} \int_0^{L/2} v^2 dv \\ &= \frac{1}{48} \frac{P^2 L^3}{EI} \end{aligned}$$

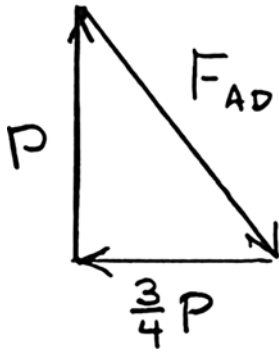
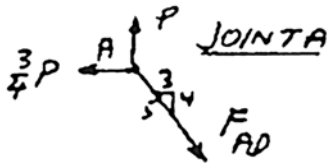
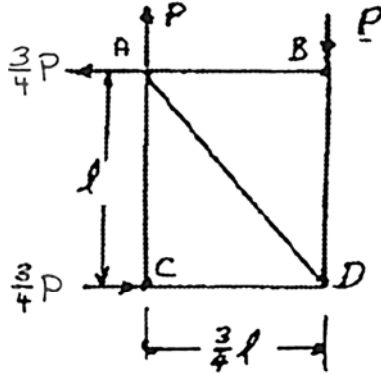
Total:

$$U = U_{AB} + U_{BC} = \frac{3}{32} \frac{P^2 L^3}{EI}$$

$$\frac{1}{2} P \delta_C = 0 \quad \delta_C = \frac{2U}{P}$$

$$\delta_C = \frac{3PL^3}{16EI} \downarrow \blacktriangleleft$$

SOLUTION 11.71



Reactions: $C = \frac{3}{4}P \rightarrow, A_x = \frac{3}{4}P \leftarrow, A_y = P \uparrow$

Members AB and AC are zero force members.

$$F_{BD} = -P$$

$$F_{CD} = \frac{3}{4}P$$

Equilibrium of joint A.

Using the force triangle,

$$F_{AD}^2 = P^2 + \left(\frac{3}{4}P\right)^2 = \frac{25}{16}P^2$$

$$F_{AD} = \frac{5}{4}P \quad (\text{tension})$$

Strain energy:

$$U = \sum \frac{F_i^2 L_i}{2EA}$$

| | F_i | L_i | $F_i^2 L_i$ |
|------|-----------------|-------------------|-----------------|
| AD | $+\frac{5}{4}P$ | $\frac{5}{4}\ell$ | $1.9531P^2\ell$ |
| BD | $-P$ | ℓ | $P^2\ell$ |
| CD | $-\frac{3}{4}P$ | $\frac{3}{4}\ell$ | $0.4219P^2\ell$ |

$$\Sigma = 3.375P^2\ell$$

$$U = \frac{3.375P^2\ell}{2EA}$$

$$\frac{1}{2}Py_B = U \quad y_B = \frac{2U}{P}$$

Deflection at B.

$$y_B = 3.375 \frac{P\ell}{EA} \quad \blacktriangleleft$$

SOLUTION 11.92

Units: Forces in kips; lengths in ft.

$$E = 29 \times 10^3 \text{ ksi} \quad I = 291 \text{ in}^4$$

$$EI = (29 \times 10^3)(291) = 8.439 \times 10^6 \text{ kip} \cdot \text{in}^2 = 58.604 \times 10^3 \text{ kip} \cdot \text{ft}^2$$

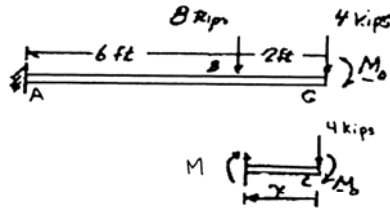
Add clockwise couple M_0 at end C.

Strain energy:
$$U = \int \frac{M^2}{2EI} dx = \frac{1}{2EI} \int M^2 dx$$

Slope at C (formula).
$$\theta_C = \frac{\partial U}{\partial M_0} = \frac{1}{EI} \int M \frac{\partial M}{\partial M_0} dx$$

Bending moment diagram.

Portion BC: ($0 \leq x \leq 2 \text{ ft}$)

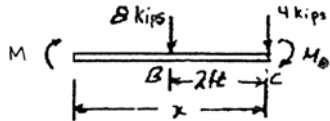


$$M = -4x - M_0$$

$$\frac{\partial M}{\partial M_0} = -1; \quad \text{Set } M_0 = 0.$$

$$\int_0^2 M \frac{\partial M}{\partial M_0} dx = \int_0^2 (-4x)(-1) dx = \frac{4}{2} x^2 \Big|_0^2 = 8 \text{ kip} \cdot \text{ft}^2$$

Portion AB: ($2 \text{ ft} \leq x \leq 8 \text{ ft}$)



$$M = -8(x - 2) - 4x - M_0$$

$$\frac{\partial M}{\partial M_0} = -1; \quad \text{Set } M_0 = 0.$$

$$\begin{aligned} \int_2^8 M \frac{\partial M}{\partial M_0} dx &= \int_2^8 [-8(x - 2 - 4x)](-1) dx \\ &= \frac{8}{2} (x - 2)^2 \Big|_2^8 + \frac{4}{2} x^2 \Big|_2^8 = (4)(6)^2 + (2)(64 - 4) = 264 \text{ kip} \cdot \text{ft}^2 \end{aligned}$$

Slope at end C. (calculated)

$$\theta_C = \frac{8 + 264}{EI} = \frac{272}{58.604 \times 10^3} \quad \theta_C = 4.64 \times 10^{-3} \text{ rad} \quad \swarrow \blacktriangleleft$$

SOLUTION 11.93

SOLUTION

Units: Forces in kips; lengths in ft.

$$E = 29 \times 10^3 \text{ ksi} \quad I = 291 \text{ in}^4$$

$$EI = (29 \times 10^3)(291) = 8.439 \times 10^6 \text{ kip} \cdot \text{in}^2 = 58.604 \times 10^3 \text{ kip} \cdot \text{ft}^2$$

Let Q be the force at end C . It is later set equal to 4 kips.

Strain energy:
$$U = \int \frac{M^2}{2EI} dx = \frac{1}{2EI} \int M^2 dx$$

Deflection at C (formula).

$$\delta_C = \frac{\partial U}{\partial Q} = \frac{1}{EI} \int M \frac{\partial M}{\partial Q} dx$$

Bending moment diagram.

Portion BC : ($0 \leq x \leq 2$ ft)

$$M = -Qx$$

$$\frac{\partial M}{\partial Q} = -x \quad \text{Set } Q = 4 \text{ kips.}$$

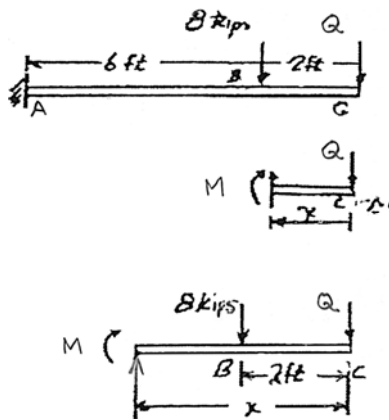
$$\begin{aligned} \int_0^2 M \frac{\partial M}{\partial Q} dx &= \int_0^2 (-4x)(-x) dx \\ &= \frac{4}{3} x^3 \Big|_0^2 = 10.67 \text{ kip} \cdot \text{ft}^3 \end{aligned}$$

Portion AB : ($2 \text{ ft} \leq x \leq 8 \text{ ft}$)

$$M = -8(x - 2) - Qx$$

$$\frac{\partial M}{\partial Q} = -x \quad \text{Set } Q = 4 \text{ kips.}$$

$$\begin{aligned} \int_2^8 M \frac{\partial M}{\partial Q} dx &= \int_2^8 [-8(x - 2) - 4x](-x) dx \\ &= 8 \int_2^8 (x^2 - 2x) dx + 4 \int_2^8 x^2 dx \\ &= 8 \left[\frac{x^3}{3} \Big|_2^8 - 16 \frac{x^2}{2} \Big|_2^8 + 4 \frac{x^3}{3} \Big|_2^8 \right] \\ &= \frac{4096}{3} - \frac{64}{3} - \frac{1024}{2} + \frac{64}{2} + \frac{2048}{3} - \frac{32}{3} \\ &= 1536 \text{ kip} \cdot \text{ft}^3 \end{aligned}$$



Deflection at C. (calculated)

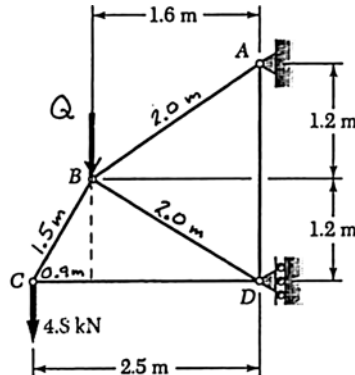
$$\delta_C = \frac{1536 + 10.67}{EI} = \frac{1546.67}{58.604 \times 10^3} = 0.0264 \text{ ft}$$

$$\delta_C = 0.317 \text{ in. } \downarrow \blacktriangleleft$$

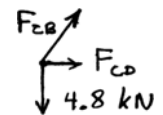
SOLUTION 11.101

Find the length of each member as shown.

Add vertical force Q at joint B .



$$\delta_B = \frac{\partial U}{\partial Q} = \frac{\partial}{\partial Q} \sum \frac{F L}{2EA} = \frac{1}{EA} \sum F \frac{\partial F}{\partial Q} L$$



Joint C:

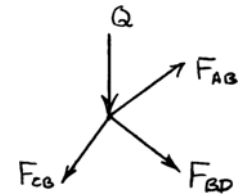
$$+\uparrow \Sigma F_y = 0: \frac{4}{5} F_{CB} - 4.8 = 0 \quad F_{CB} = 6.0 \text{ kN}$$

$$+\rightarrow \Sigma F_x = 0: \frac{3}{5} F_{CB} + F_{CD} = 0 \quad F_{CD} = -3.6 \text{ kN}$$

Joint B:

$$+\rightarrow \Sigma F_x = 0: \frac{4}{5} F_{AB} + \frac{4}{5} F_{BD} - 3.6 = 0$$

$$+\uparrow \Sigma F_y = 0: \frac{3}{5} F_{AB} - \frac{3}{5} F_{BD} - 4.8 - Q = 0$$



Solving simultaneously,

$$F_{AB} = 6.25 + 0.8333Q \text{ kN}$$

$$F_{BD} = -1.75 - 0.8333Q \text{ kN}$$

Joint D:

$$+\uparrow \Sigma F_y = 0: \frac{3}{5} F_{BD} + F_{AD} = 0$$

$$F_{AD} = -\frac{3}{5} F_{BD} = 1.05 + 0.5Q$$



| Member | $F(10^3 \text{ N})$ | $\partial F / \partial Q$ | $L(\text{ m})$ | with $Q = 0$ $F(\partial F / \partial Q)L(10^3 \text{ N} \cdot \text{ m})$ |
|----------|---------------------|---------------------------|----------------|---|
| AB | $6.25 + 0.8333Q$ | 0.8333 | 2.0 | 10.4167 |
| AD | $1.05 + 0.5Q$ | 0.5 | 2.4 | 1.26 |
| BD | $-1.75 - 0.8333Q$ | -0.8333 | 2.0 | 2.9167 |
| BC | 6.0 | 0 | 1.5 | 0 |
| CD | -3.6 | 0 | 2.5 | 0 |
| Σ | | | | 14.593 |

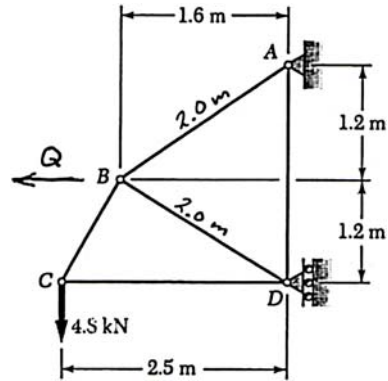
$$\begin{aligned}\delta_B &= \frac{1}{EA} \Sigma F(\partial F / \partial Q)L \\ &= \frac{14.593 \times 10^3}{(200 \times 10^9)(500 \times 10^{-6})} \\ &= 145.9 \times 10^{-6} \text{ m}\end{aligned}$$

$$\delta_B = 0.1459 \text{ mm} \downarrow \blacktriangleleft$$

SOLUTION 11.102

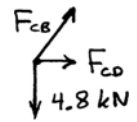
Find the length of each member as shown.

Add horizontal force Q at joint B .



$$\delta_B = \frac{\partial U}{\partial Q} = \frac{\partial}{\partial Q} \sum \frac{F^2 L}{2EA} = \frac{1}{EA} \sum F \frac{\partial F}{\partial Q} L$$

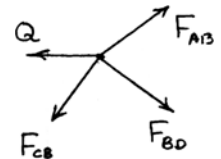
Joint C:
$$+\uparrow \Sigma F_y = 0: \frac{4}{5} F_{CB} - 4.8 = 0 \quad F_{CB} = 6.0 \text{ kN}$$



Joint B:
$$+\rightarrow \Sigma F_x = 0: \frac{3}{5} F_{CB} + F_{CD} = 0 \quad F_{CD} = -3.6 \text{ kN}$$

$$+\rightarrow \Sigma F_x = 0: \frac{4}{5} F_{AB} + \frac{4}{5} F_{BD} - 3.6 - Q = 0$$

$$+\uparrow \Sigma F_y = 0: \frac{3}{5} F_{AB} - \frac{3}{5} F_{BD} - 4.8 = 0$$



Solving simultaneously,
$$F_{AB} = 6.25 + 0.625Q \text{ kN}$$

$$F_{BD} = -1.75 + 0.625Q \text{ kN}$$

Joint D:
$$+\uparrow \Sigma F_y = 0: \frac{3}{5} F_{BD} + F_{AD} = 0$$

$$F_{AD} = -\frac{3}{5} F_{BD} = 1.05 - 0.375Q$$



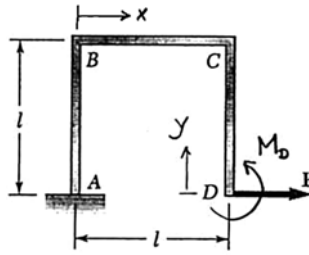
| Member | $F(10^3 \text{ N})$ | $\partial F/\partial Q$ | $L(\text{m})$ | $F(\partial F/\partial Q)L$ ($10^3 \text{ N} \cdot \text{m}$) |
|----------|---------------------|-------------------------|---------------|--|
| AB | $6.25 + 0.625Q$ | 0.625 | 2.0 | 7.8125 |
| AD | $1.05 + 0.375Q$ | -0.375 | 2.4 | -0.9450 |
| BD | $-1.75 + 0.625Q$ | 0.625 | 2.0 | -2.1875 |
| BC | 6.0 | 0 | 1.5 | 0 |
| CD | -3.6 | 0 | 2.5 | 0 |
| Σ | | | | 4.680 |

$$\begin{aligned}
 \delta_B &= \frac{1}{EA} \Sigma F(\partial F/\partial Q)L \\
 &= \frac{4.680 \times 10^3}{(200 \times 10^9)(500 \times 10^{-6})} \\
 &= 46.8 \times 10^{-6} \text{ m}
 \end{aligned}$$

$$\delta_B = 0.0468 \text{ mm} \leftarrow \blacktriangleleft$$

SOLUTION 11.106

Add couple M_D at point D .



Reactions at A:

$$R_{Ay} = 0, \quad R_{Ax} = P \leftarrow, \quad M_A = M_0 \curvearrowleft$$

Member AB:

$$M = M_A + R_{Ay}y = M_D + Py \quad \frac{\partial M}{\partial P} = y, \quad \frac{\partial M}{\partial M_D} = 1$$

$$U_{AB} = \int_0^l \frac{M^2}{2EI} dy$$

Set $M_D = 0$.

$$\frac{\partial U_{AB}}{\partial P} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial P} dy = \frac{1}{EI} \int_0^l (Py)y dy = \frac{Pl^3}{3EI}$$

$$\frac{\partial U_{AB}}{\partial M_0} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial M_0} dy = \frac{1}{EI} \int_0^l (Py)(1) dy = \frac{Pl^2}{2EI}$$

Member BC:

$$M = M_A + R_{Ax}l = M_D + Pl \quad \frac{\partial M}{\partial P} = l, \quad \frac{\partial M}{\partial M_D} = 1$$

$$U_{BC} = \int_0^l \frac{M^2}{2EI} dx$$

Set $M_D = 0$.

$$\frac{\partial U_{BC}}{\partial P} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial P} dx = \frac{1}{EI} \int_0^l (Pl)(l) dx = \frac{Pl^3}{EI}$$

$$\frac{\partial U_{BC}}{\partial M_D} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial M_D} dx = \frac{1}{EI} \int_0^l (Pl)(1) dx = \frac{Pl^2}{EI}$$

Member CD:

$$M = M_D + Py \quad \frac{\partial M}{\partial P} = y \quad \frac{\partial M}{\partial M_D} = 1$$

$$U_{CD} = \int_0^l \frac{M^2}{2EI} dy$$

Set $M_D = 0$.

$$\frac{\partial U_{CD}}{\partial P} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial P} dy = \frac{1}{EI} \int_0^l (Py)(y) dy = \frac{Pl^3}{3EI}$$

$$\frac{\partial U_{CD}}{\partial M_D} = \frac{1}{EI} \int_0^l M \frac{\partial M}{\partial M_D} dy = \frac{1}{EI} \int_0^l (Py)(1) dy = \frac{Pl^2}{2EI}$$

(a) Horizontal deflection of point D.

$$\delta_P = \frac{\partial U_{AB}}{\partial P} + \frac{\partial U_{BC}}{\partial P} + \frac{\partial U_{CD}}{\partial P} = \left(\frac{1}{3} + 1 + \frac{1}{3} \right) \frac{Pl^3}{EI}$$

$$\delta_P = \frac{5Pl^3}{3EI} \rightarrow \blacktriangleleft$$

(b) Slope at point D.

$$\theta_D = \frac{\partial U_{AB}}{\partial M_D} + \frac{\partial U_{BC}}{\partial M_D} + \frac{\partial U_{CD}}{\partial M_D} = \left(\frac{1}{2} + 1 + \frac{1}{2} \right) \frac{Pl^2}{EI}$$

$$\theta_D = \frac{2Pl^2}{EI} \curvearrowright \blacktriangleleft$$

SOLUTION 11.108

Add horizontal force Q at point A.

Over AB:

$$M = \frac{1}{2}Pv + \frac{\sqrt{3}}{2}Qv$$

$$\frac{\partial M}{\partial P} = \frac{1}{2}v \quad \frac{\partial M}{\partial Q} = \frac{\sqrt{3}}{2}v$$

$$U_{AB} = \int_0^L \frac{M^2}{2EI} dx$$

Set $Q = 0$.

$$\frac{\partial U_{AB}}{\partial P} = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial P} dv = \frac{1}{EI} \int_0^L \left(\frac{1}{2}Pv \right) \left(\frac{1}{2}v \right) dv$$

$$= \frac{1}{12} \frac{PL^3}{EI}$$

$$\frac{\partial U_{AB}}{\partial Q} = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial Q} dv = \frac{1}{EI} \int_0^L \left(\frac{1}{2}Pv \right) \frac{\sqrt{3}}{2} dv = \frac{\sqrt{3}}{12} \frac{PL^3}{EI}$$

Over BC:

$$M = -P \left(x - \frac{L}{2} \right) + \frac{\sqrt{3}}{2}QL, \quad \frac{\partial M}{\partial P} = - \left(x - \frac{L}{2} \right), \quad \frac{\partial M}{\partial Q} = \frac{\sqrt{3}}{2}L$$

$$U_{BC} = \int_0^L \frac{M^2}{2EI} dx$$

Set $Q = 0$.

$$\frac{\partial U_{BC}}{\partial P} = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial P} dx = \frac{1}{EI} \int_0^L P \left(x - \frac{L}{2} \right)^2 dx = \frac{P}{3EI} \left(x - \frac{L}{2} \right)^3 \Big|_0^L = \frac{1}{12} \frac{PL^3}{EI}$$

$$\frac{\partial U_{BC}}{\partial Q} = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial Q} dx = \frac{-1}{EI} \int_0^L P \left(x - \frac{L}{2} \right) \left(\frac{\sqrt{3}}{2} \right) L dx = -\frac{\sqrt{3}P}{4EI} \left(x - \frac{L}{2} \right)^2 \Big|_0^L = 0$$

(a) Vertical deflection of point A.

$$\delta_P = \frac{\partial U_{AB}}{\partial P} + \frac{\partial U_{BC}}{\partial P} \qquad \delta_P = \frac{PL^3}{6EI} \downarrow \blacktriangleleft$$

(b) Horizontal deflection of point A.

$$\delta_Q = \frac{\partial U_{AB}}{\partial Q} + \frac{\partial U_{BC}}{\partial Q} = \frac{\sqrt{3}}{12} \frac{PL^3}{EI} \qquad \delta_Q = 0.1443 \frac{PL^3}{EI} \rightarrow \blacktriangleleft$$

