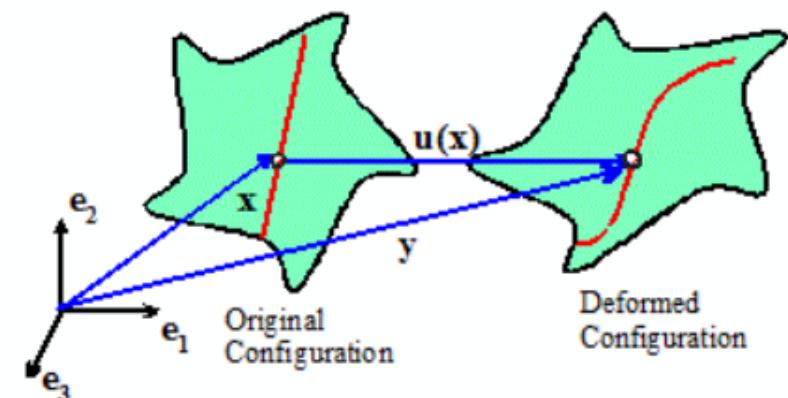


Infinitesimal Deformations

$$\underline{u} : \nabla \underline{u} : \nabla \underline{u} \ll 1$$

This requires both small strains
and small rotations



Infinitesimal Strain Tensor

Properties :

$$\textcircled{1} \quad \boldsymbol{\varepsilon} \propto \boldsymbol{\epsilon}$$

$$\text{To see this note } \boldsymbol{\epsilon} = \frac{1}{2} (\boldsymbol{F}^T \boldsymbol{F} - \boldsymbol{I})$$

$$= \frac{1}{2} [(\boldsymbol{I} + \nabla \underline{u})^T (\boldsymbol{I} + \nabla \underline{u}) - \boldsymbol{I}]$$

$$= \boldsymbol{\epsilon} + \frac{1}{2} \nabla \underline{u}^T \nabla \underline{u}$$

neglect

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$$\textcircled{2} \quad \text{trace } (\varepsilon) = \frac{dV - dV_0}{dV_0}$$

To see this recall $\frac{dV}{dV_0} = \det(I + \nabla u)$

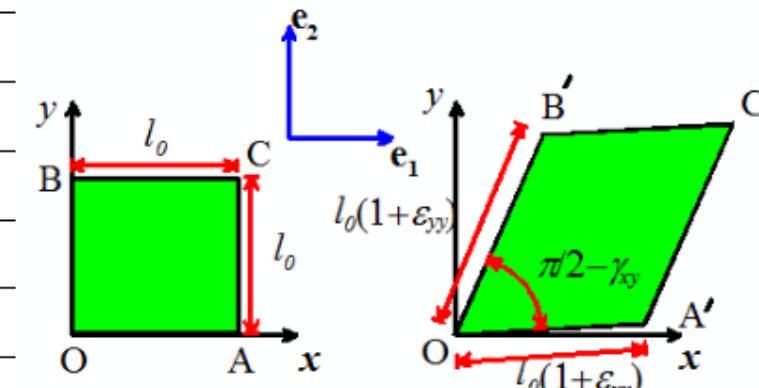
$$= 1 + \text{tr}(Du) + O((Du)^2)$$

$$= 1 + \text{tr}(\varepsilon) + \dots$$

$$\frac{dV - 1}{dV_0} = \text{tr } (\varepsilon)$$

(3) Components of ε related to length & angle changes

$$\varepsilon = \begin{bmatrix} \varepsilon_{xx} & \varepsilon_{xy} \\ \varepsilon_{xy} & \varepsilon_{yy} \end{bmatrix}$$



$$\gamma_{xy} = 2\varepsilon_{xy} \leftarrow \text{"Engineering" strain}$$

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$$\textcircled{4} \quad \frac{l - l_0}{l_0} \approx m \cdot \underline{\epsilon_m} \\ \approx n \cdot \underline{\epsilon_n}$$

(5) $\underline{\epsilon}$ is symmetric

\Rightarrow real eigenvalues
- principal strains

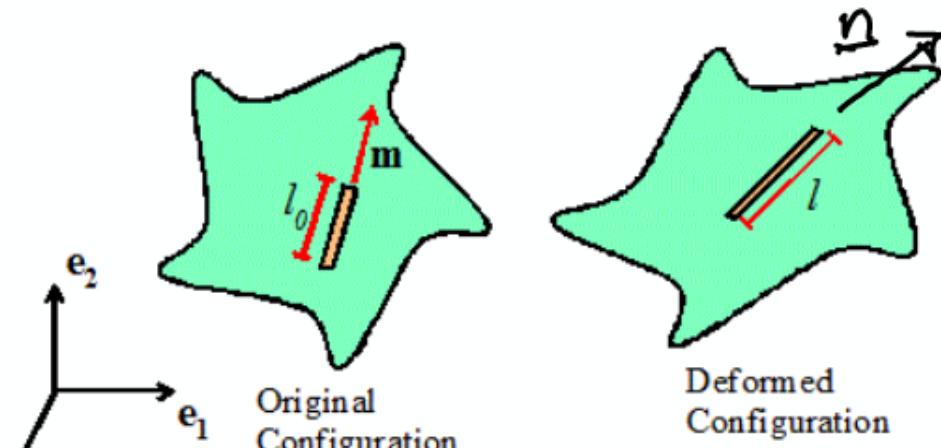
Orthogonal eigenvectors : principal strain directions

Infinitesimal rotation tensor

$$w = \text{skew}(D\underline{u}) \\ w_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} - \frac{\partial u_j}{\partial x_i} \right)$$

Properties :

$$\textcircled{1} \quad d\underline{x} \cdot (I + w)d\underline{x} = d\underline{x} \cdot d\underline{x}$$

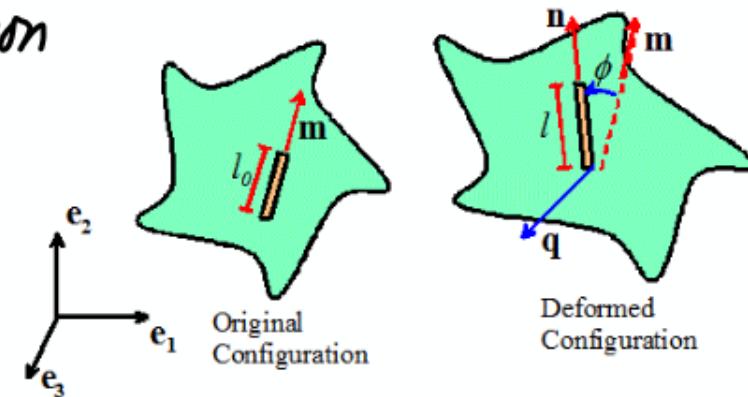


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Hence \underline{W} represents an infinitesimal rotation

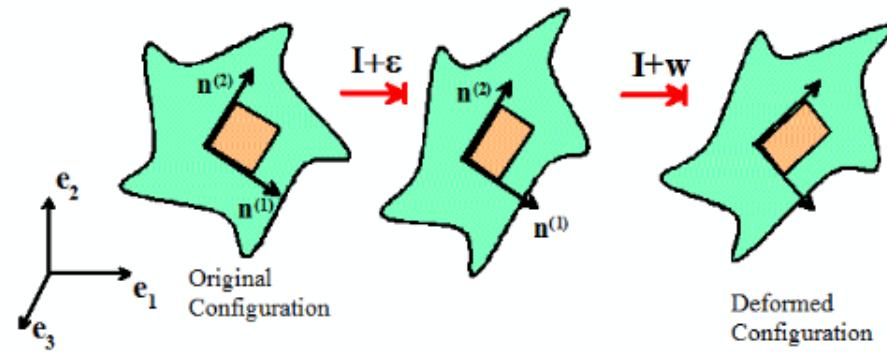
$$w \underline{dx} = \sin \theta \underline{n} \times \underline{dx} \quad \sin \theta \underline{n} = \text{dual}(\underline{w})$$

θ represents small angle of rotation
of average rotation of
all fibers passing through
a point



Decomposition of $\nabla \underline{u}$

$$\nabla \underline{u} = \underline{\epsilon} + \underline{N}$$



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Inverting the strain-displacement relations

Given $\epsilon_{ij}(\underline{x})$, can we find $u_i(\underline{x})$?

① Rigid motions have $\epsilon = 0 \Rightarrow$ can only find u to within a rigid motion

② 6 ϵ_{ij} , but only 3 $u_i \Rightarrow \epsilon_{ij}$ are not all independent

Compatibility conditions

$$\epsilon = \text{sym } (\nabla u) \Leftrightarrow \nabla \times \nabla \times \epsilon = 0$$

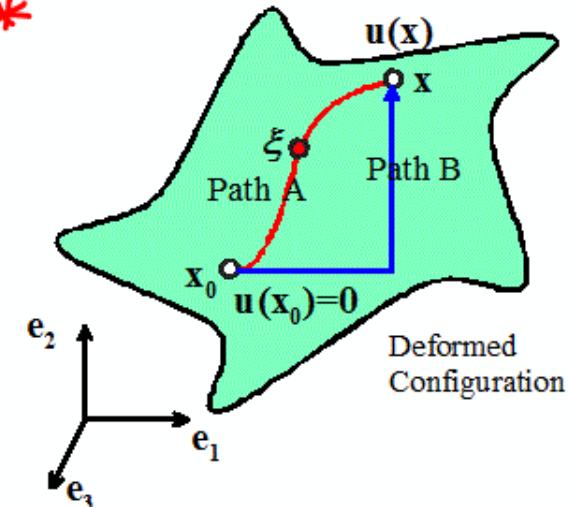
$$\text{or } \frac{\partial^2 \epsilon_{ij}}{\partial x_k \partial x_l} + \frac{\partial^2 \epsilon_{ke}}{\partial x_i \partial x_j} - \frac{\partial^2 \epsilon_{il}}{\partial x_j \partial x_k} - \frac{\partial^2 \epsilon_{ik}}{\partial x_j \partial x_l} = 0$$

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$$\text{Also } u_i = \int_{\underline{x}_0}^{\underline{x}} \Lambda_{ij}(\underline{x}, \underline{\xi}) d\xi_j *$$

$$\Lambda_{ij} = \varepsilon_{ij} + (x_k - \xi_k) \left(\frac{\partial \varepsilon_{ij}}{\partial \xi_k} - \frac{\partial \varepsilon_{kj}}{\partial \xi_i} \right)$$

We need to show integral is path independent



For this to be true $\Lambda_{ij} d\xi_j$ must be a perfect differential

$$\Rightarrow \Lambda_{ij} = \frac{\partial P_i}{\partial \xi_j} \text{ for some } P_i$$

$$\Rightarrow \frac{\partial \Lambda_{ij}}{\partial \xi_n} = \frac{\partial^2 P_i}{\partial \xi_j \partial \xi_n} = \frac{\partial \Lambda_{in}}{\partial \xi_j} \Rightarrow \frac{\partial \Lambda_{ij}}{\partial \xi_n} - \frac{\partial \Lambda_{in}}{\partial \xi_j} = 0$$

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$$\Lambda_{ij} = \varepsilon_{ij} + (x_k - \xi_k) \left(\frac{\partial \varepsilon_{ij}}{\partial \xi_k} - \frac{\partial \varepsilon_{kj}}{\partial \xi_i} \right)$$

$$\frac{\partial \Lambda_{ij}}{\partial \xi_n} = \frac{\partial \varepsilon_{ij}}{\partial \xi_n} - \left(\frac{\partial \varepsilon_{ij}}{\partial \xi_n} - \frac{\partial \varepsilon_{nj}}{\partial \xi_i} \right) + (x_k - \xi_k) \left(\frac{\partial \varepsilon_{ij}}{\partial \xi_n \partial \xi_k} - \frac{\partial \varepsilon_{kj}}{\partial \xi_n \partial \xi_i} \right)$$

$$\frac{\partial \Lambda_{in}}{\partial \xi_j} = \frac{\partial \varepsilon_{in}}{\partial \xi_j} - \left(\frac{\partial \varepsilon_{in}}{\partial \xi_j} - \frac{\partial \varepsilon_{nj}}{\partial \xi_i} \right) + (x_k - \xi_k) \left(\frac{\partial \varepsilon_{in}}{\partial \xi_j \partial \xi_k} - \frac{\partial \varepsilon_{kn}}{\partial \xi_j \partial \xi_i} \right)$$

$$\frac{\partial \Lambda_{ij}}{\partial \xi_n} - \frac{\partial \Lambda_{in}}{\partial \xi_j} = (x_k - \xi_k) \left(\frac{\partial \varepsilon_{ij}}{\partial \xi_n \partial \xi_k} + \frac{\partial \varepsilon_{kn}}{\partial \xi_j \partial \xi_i} - \frac{\partial \varepsilon_{kj}}{\partial \xi_n \partial \xi_i} - \frac{\partial \varepsilon_{in}}{\partial \xi_j \partial \xi_k} \right)$$

Compatibility condition!

The integral is a formal result but not very useful

- Usually use a direct integration of components instead

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page 8 Example: find the 2D displacement field generating the strain

$$\varepsilon_{\alpha\beta} = (3+\nu)a^2 \delta_{\alpha\beta} - (1+\nu)(x_\gamma x_\gamma \delta_{\alpha\beta} + 2x_\alpha x_\beta) \quad \alpha, \beta, \gamma = 1, 2$$

① $\varepsilon_{11} = \frac{\partial u_1}{\partial x_1} = (3+1\nu)a^2 - (1+\nu)(3x_1^2 + x_2^2)$
 $\Rightarrow u_1 = (3+1\nu)a^2 x_1 - (1+\nu)(x_1^3 + x_1 x_2^2) + f(x_2)$

② $\varepsilon_{22} = \frac{\partial u_2}{\partial x_2} = (3+1\nu)a^2 - (1+\nu)(3x_2^2 + x_1^2)$
 $\Rightarrow u_2 = (3+1\nu)a^2 x_2 - (1+\nu)(x_2^3 + x_2 x_1^2) + g(x_1)$

③ $2\varepsilon_{12} = \frac{\partial u_1}{\partial x_2} \rightarrow \frac{\partial u_2}{\partial x_1} = -4(1+\nu)x_1 x_2 = \frac{\partial f}{\partial x_2} + \frac{\partial g}{\partial x_1} - 4(1+\nu)x_1 x_2$
 $\Rightarrow \frac{\partial f}{\partial x_2} + \frac{\partial g}{\partial x_1} = 0 \quad \frac{\partial f}{\partial x_2} = -\omega \quad \frac{\partial g}{\partial x_1} = +\omega$
 $\omega = \text{const}$

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$$f = C_1 - \omega x_2 \quad g = C_2 + \omega x_1$$

Rigid motion:

$$\text{Hence } U_\alpha = (3+\nu) a^2 x_\alpha - (1+\nu) x_\gamma x_\beta x_\alpha \\ + C_\alpha + \epsilon_{\alpha\beta\gamma} \omega x_\beta$$

Proof that U_i in * generates correct ϵ_{ij}
will be covered in L11

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